



4. Unsinkable Disk

“A metal disk with a *hole* at its centre *sinks* in a container filled with *water*.

When a *vertical water jet* hits the *centre of the disc*, it may *float* on the water surface. *Explain* this phenomenon and investigate the *relevant parameters*.”



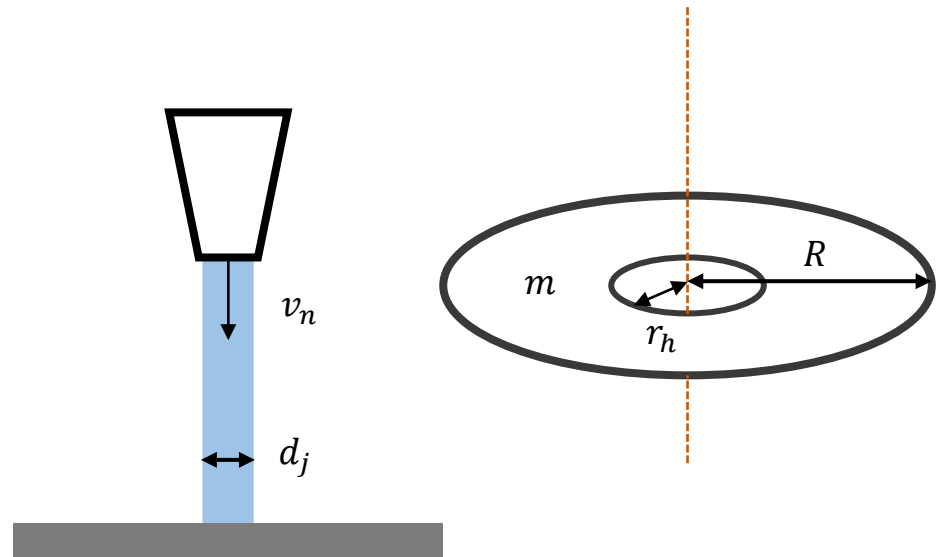
Problem Statement

“A metal disk with a *hole* at its centre *sinks* in a container filled with *water*.

When a *vertical water jet* hits the *centre of the disc*, it may *float* on the water surface. *Explain* this phenomenon and investigate the *relevant parameters*.”

Parameters:

1. Jet speed
2. Jet diameter
3. Hole radius
4. Disk radius
5. Disk mass



Overview

1

Introduction

Reproduction of the Phenomenon, Qualitative Explanation

2

Experimental Setup

Measurement Devices, Calibration, Camera Views

3

Theoretical Model

Flow Dynamics, Hydraulic Jump

4

Key Parameter Interactions

Effects of Varying Physical Parameters

5

Conclusion

Further Insights

Phenomenon Cases

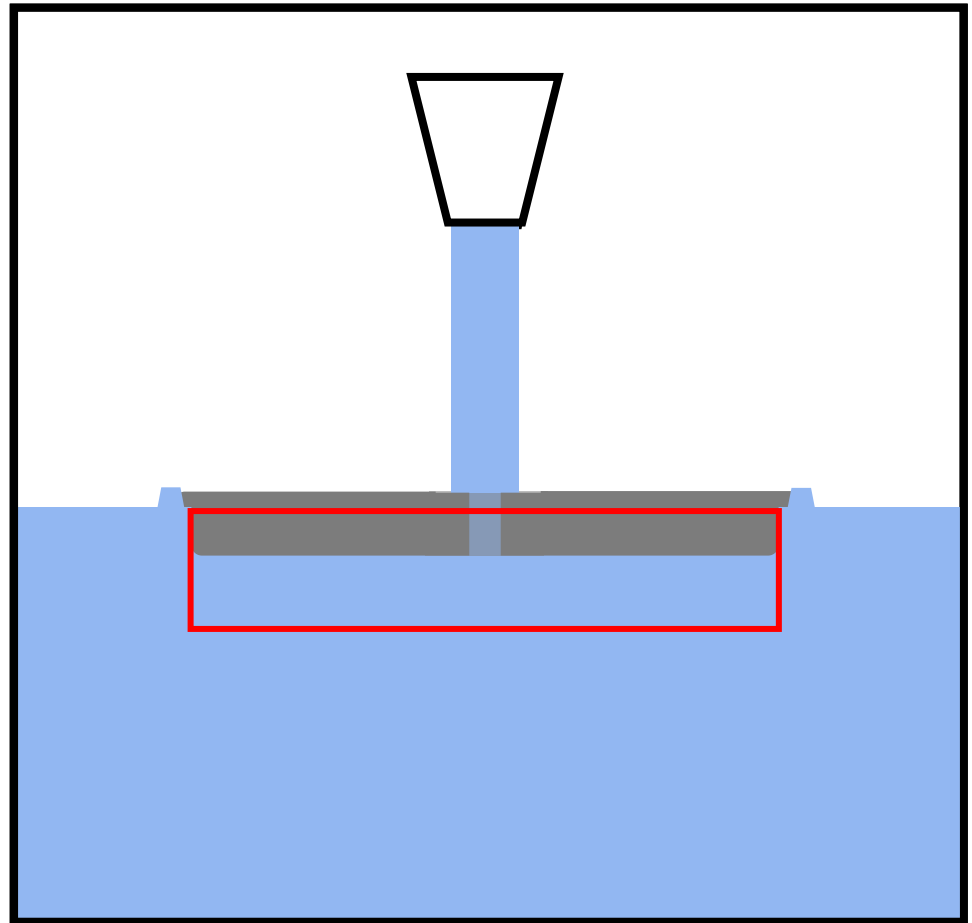
The Unsinkable Disk has
two distinct cases

Case 1:

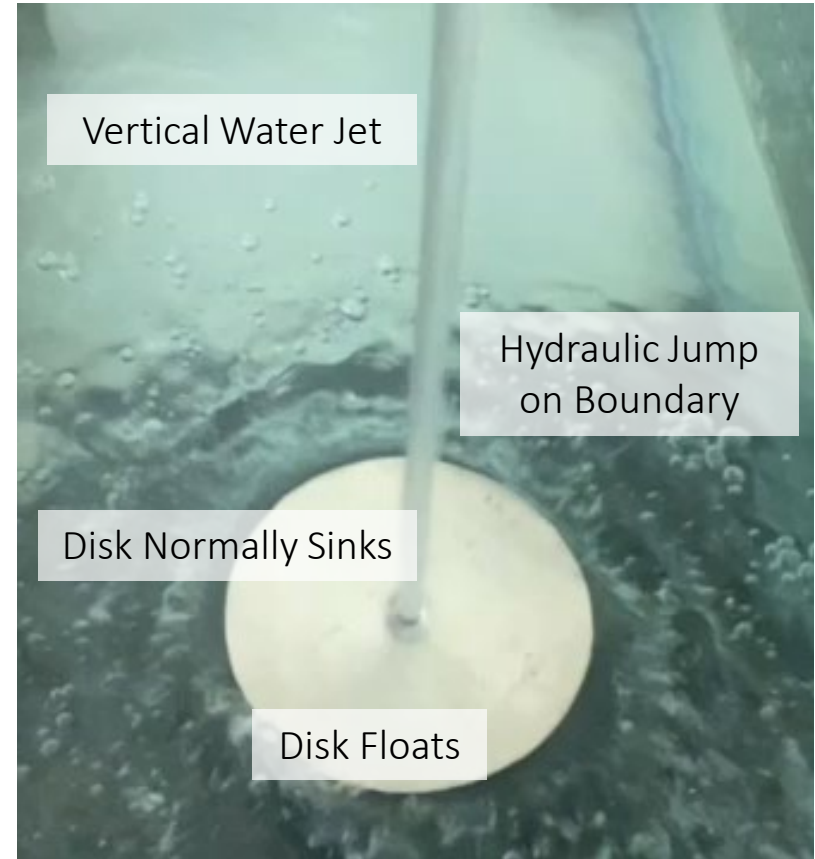
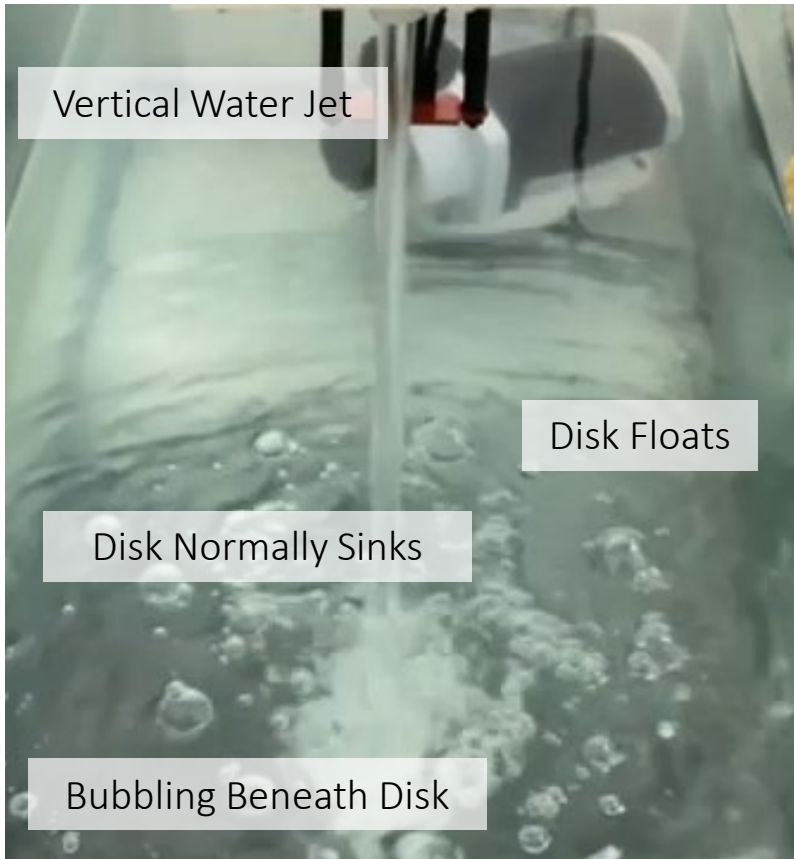
Hole Radius $>$ Jet Radius

Case 2:

Hole Radius $<$ Jet Radius



Phenomenon



Experimental Setup

Introduction

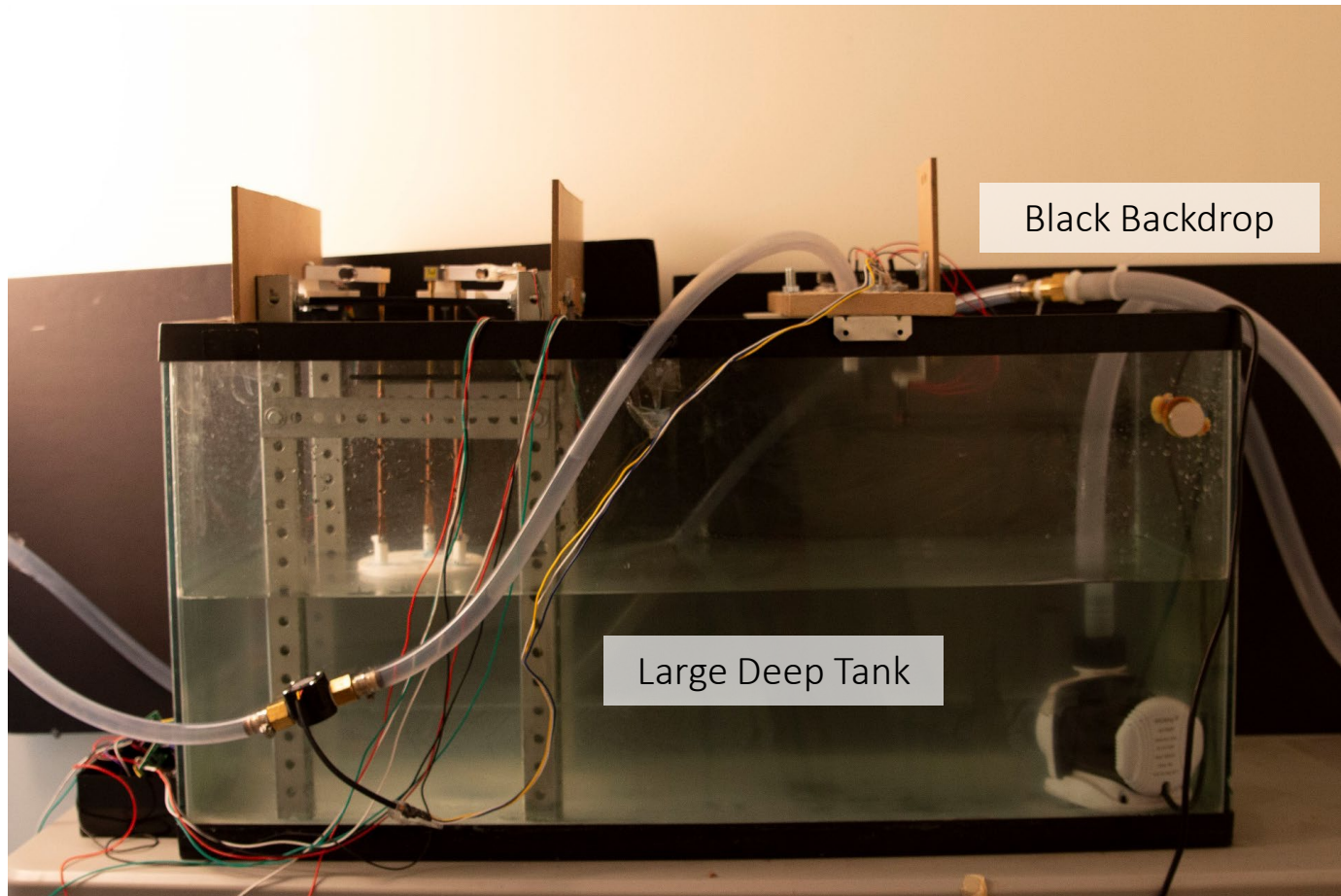
Experimental Setup

Theoretical Model

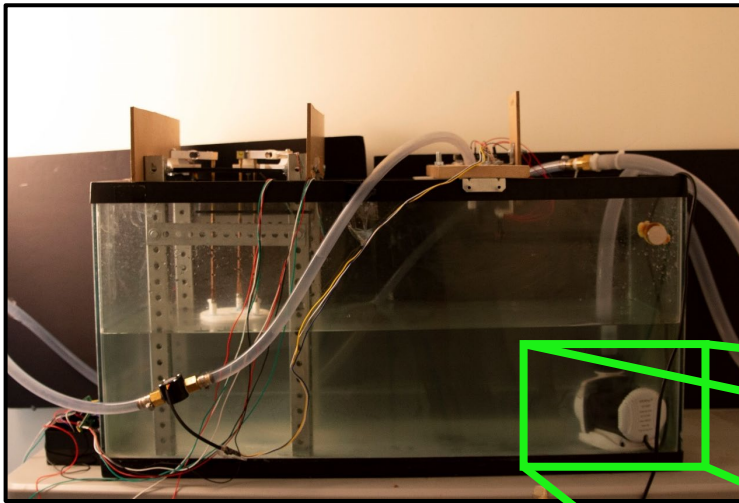
Key Parameters

Conclusion

Experimental Setup



Pump



Specifications:

DC Aquarium Pump

20 Adjustable Flow Rates With Remote

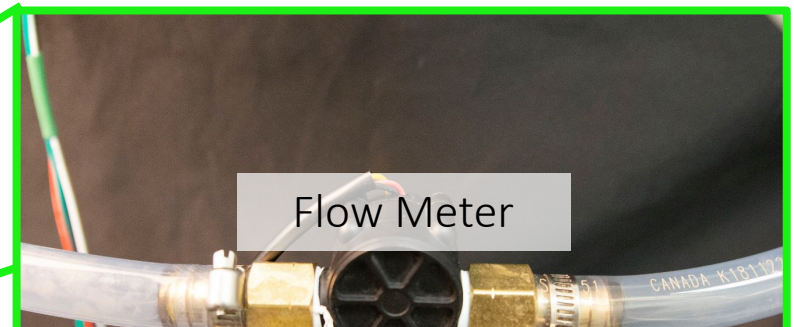
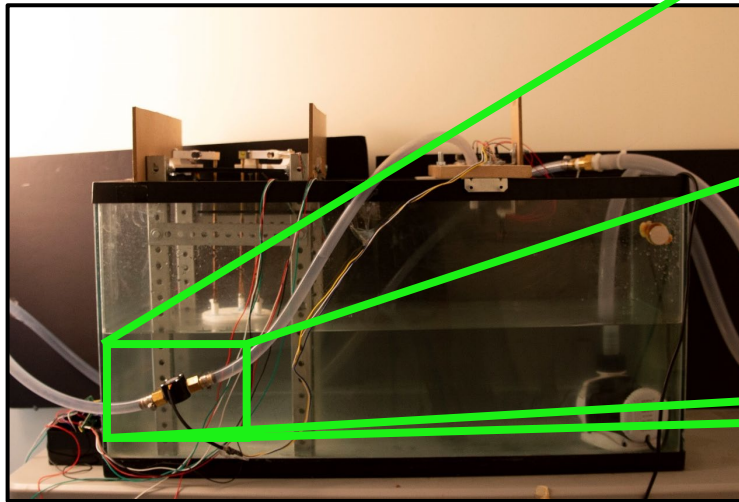
< 80mL/s with Ball valve

Up to 3260 mL/s

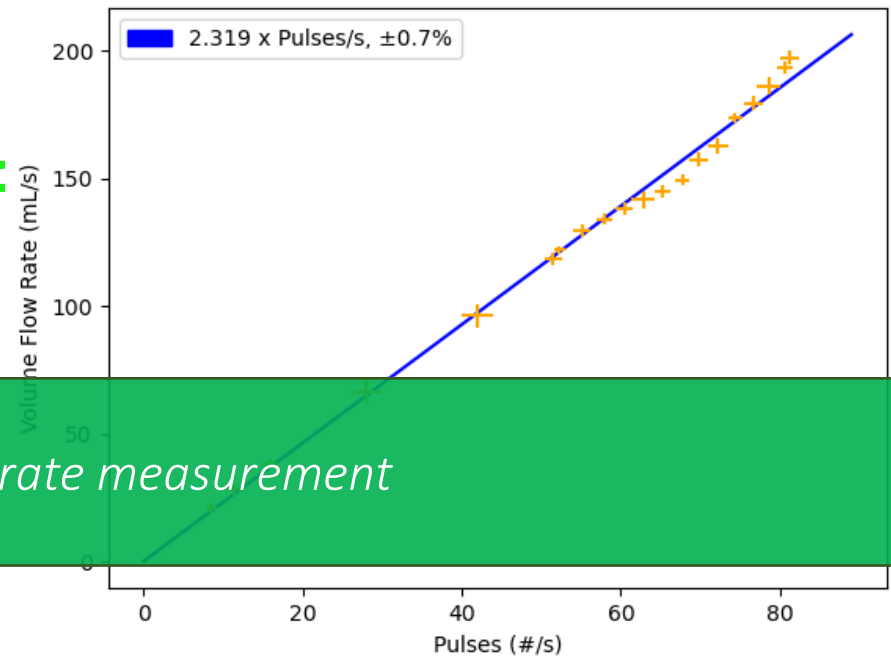
Controlled flow rate variation

Submersible Pump

Flow Meter



Volume Flow Rate vs. Pulse Count



Specifications:

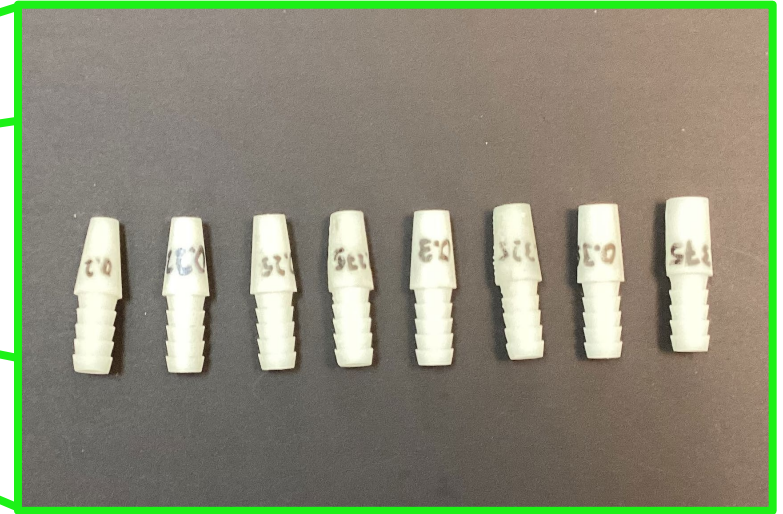
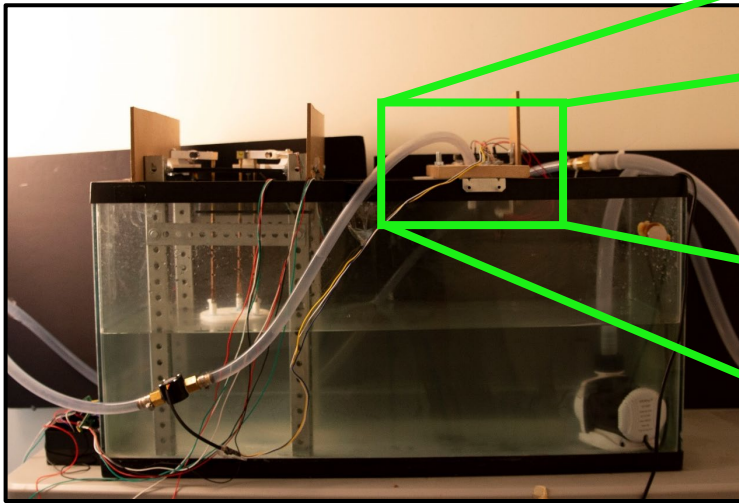
DIGITEN Water Flow Sens

17mL/s – 500mL/s

Accurate flow rate measurement

Pulse Counter with Arduino

Nozzles



Specifications:

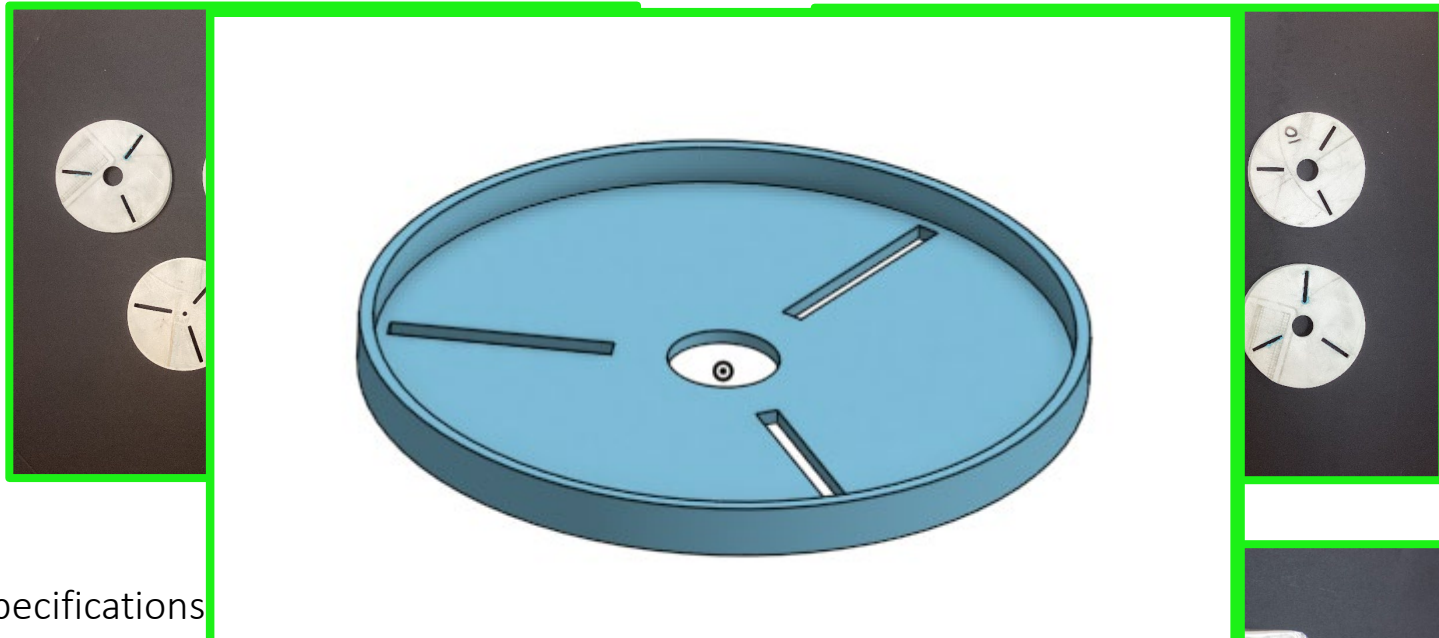


7 Different nozzle diameters *Controlled jet radius change*

SLA 3d Printer - Resin

0.375in. – 0.200in. diameter

Disks



Specifications



Controlled change of mass, radius and hole radius

FDM 3d Printer – PLA

Clay to close slits

Clay to vary disk mass

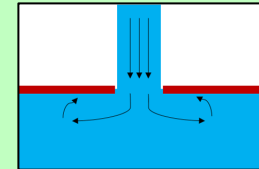
Theoretical Model

Theoretical Model

A

Flow Dynamics (hole > jet)

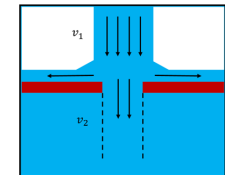
Force Analysis, Jet Effects, Empirical Model



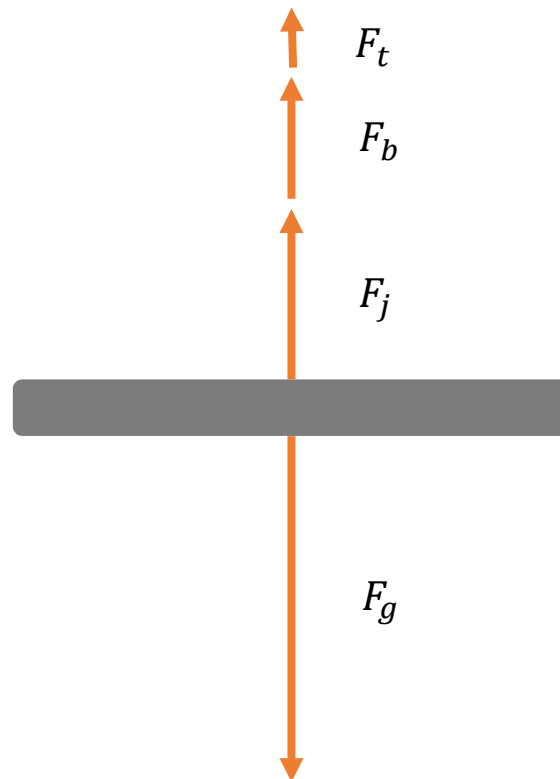
B

Hydraulic Jump (jet > hole)

Impinging Force, Archimedes' Principle, Empirical Model



Free Body Diagram



$F_g = \text{Force of gravity}$

$F_j = \text{Force of jet}$

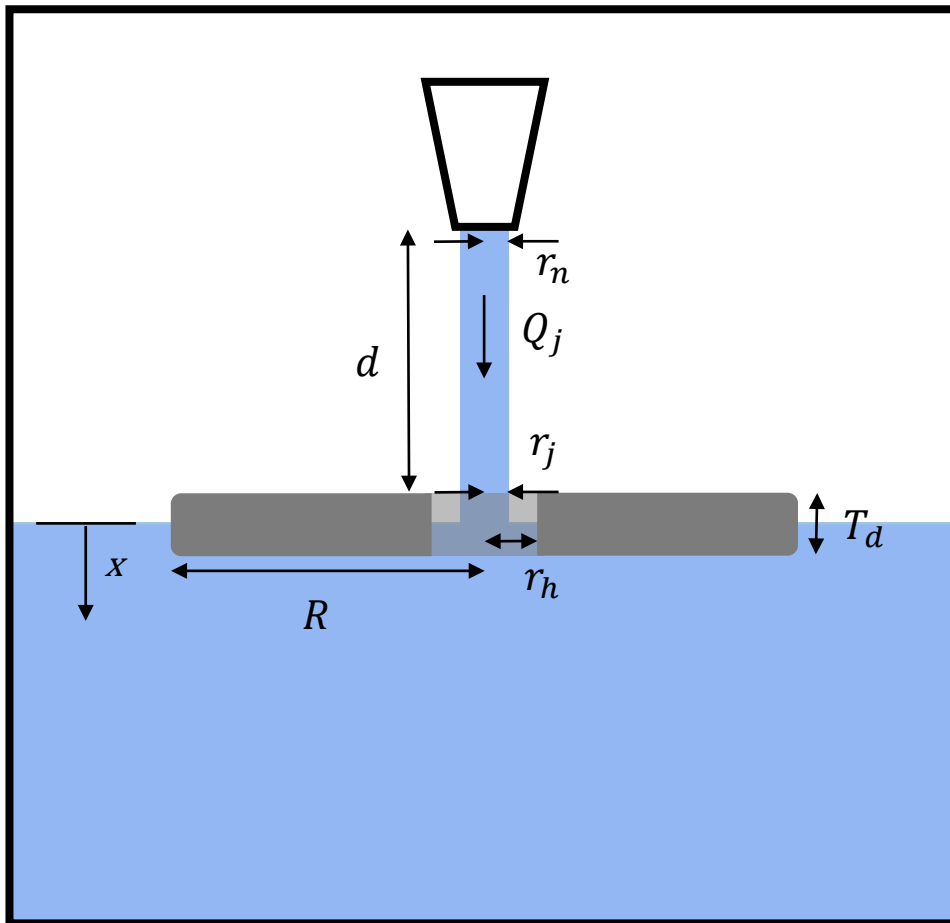
$F_b = \text{Buoyant force}$

$F_t = \text{Force of surface tension}$

Equilibrium necessary to float:

$$F_g - F_b - F_t - F_j = 0$$

Geometry



Terms:

r_n = nozzle radius

r_j = jet radius

r_h = hole radius

R = disk radius

T_d = disk thickness

d = nozzle height

Q_j = volume flow rate

Flow Dynamics

Hydraulic Jump

Assumptions



Jet does not collide with edges of disk



Jet remains vertical, centered, and at constant velocity



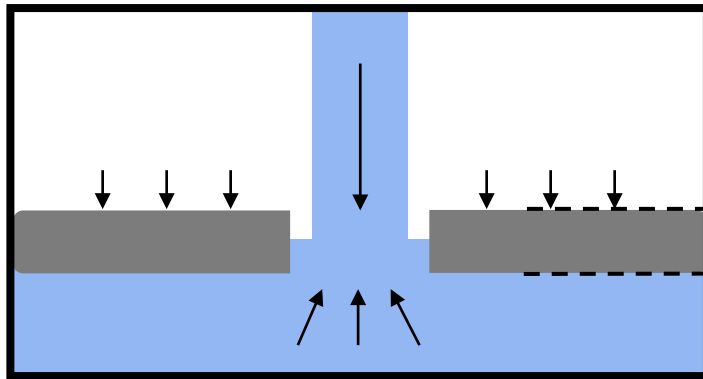
Tank is large enough to neglect wave effects



Water is incompressible

Jet Forces

Decompose jet forces into two components

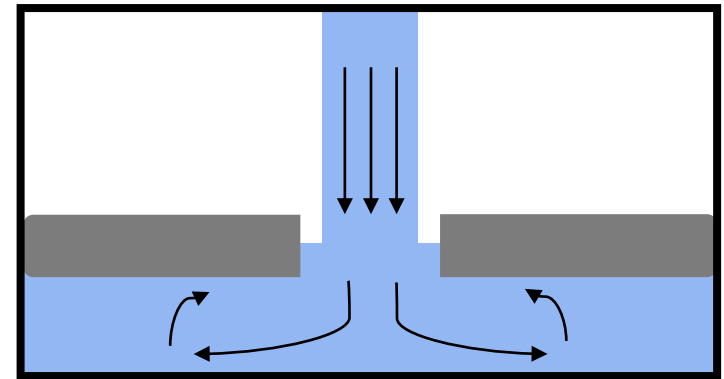


①
②

Opposing flow through hole

Through Bernoulli's equation:

$$P_{\text{atm}} + \rho g T_d = P_{\text{atm}} + \rho g x + \frac{1}{2} \rho v^2$$



Additional vortices and gas

Through continuity:

$$\pi R^2 \dot{x} = \pi r^2 v$$

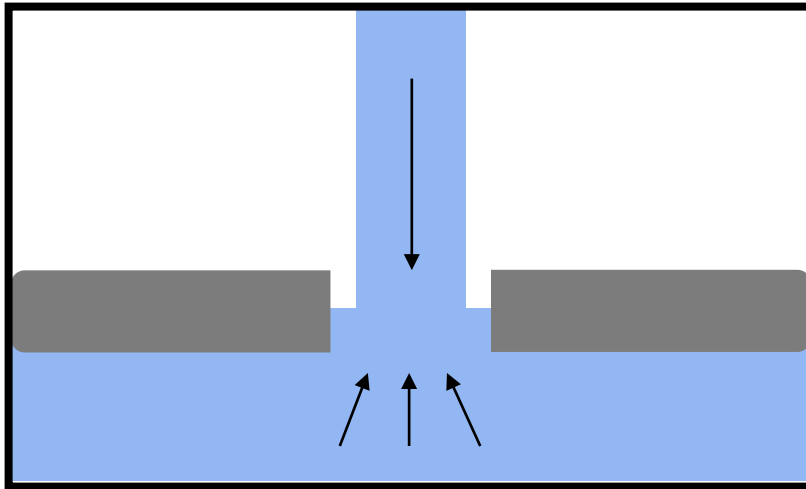
$$\dot{x} = \sqrt{2g(T_d - x)} \left(\frac{r}{R}\right)^2$$

Flow Dynamics

Hydraulic Jump

Potential Flow

For disks that barely dense enough for sinking, jet flow only opposes potential flow



$$v = \dot{x} = \sqrt{2g(T_d - x)} \left(\frac{r}{R}\right)^2$$

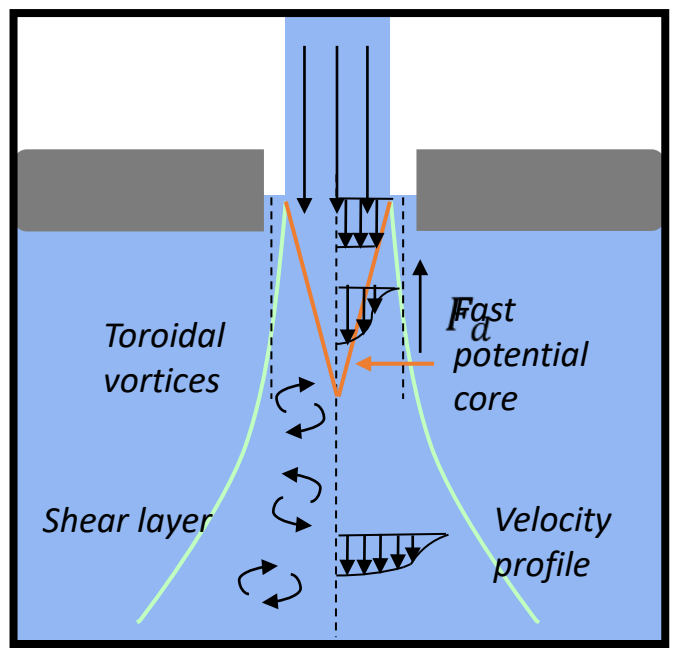
*Calculating minimum flow rate needed
from v :*

$$Q = \pi r^2 \sqrt{2g(T_d - x)} \left(\frac{r}{R}\right)^2$$



Additional jet forces must be present for heavier disks

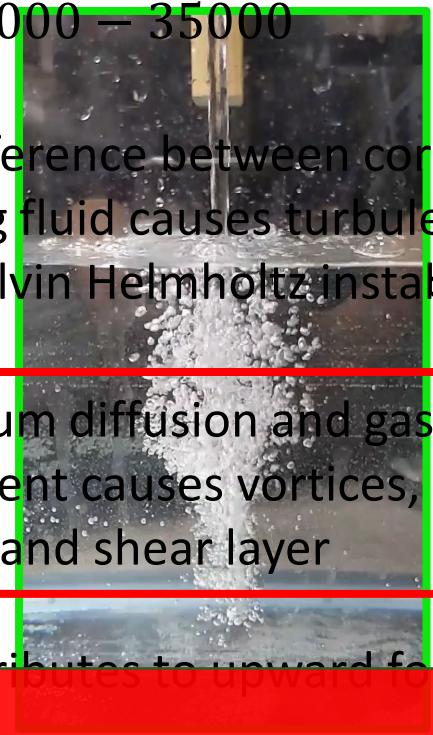
Vortices and Turbulence



We calculate:
 $Re: 25000 - 35000$

Velocity difference between core and surrounding fluid causes turbulence and vortices (Kelvin Helmholtz instability)

Momentum diffusion and gas entrainment causes vortices, bubbling and shear layer



After impact, jet continues to

flow while slowed by drag

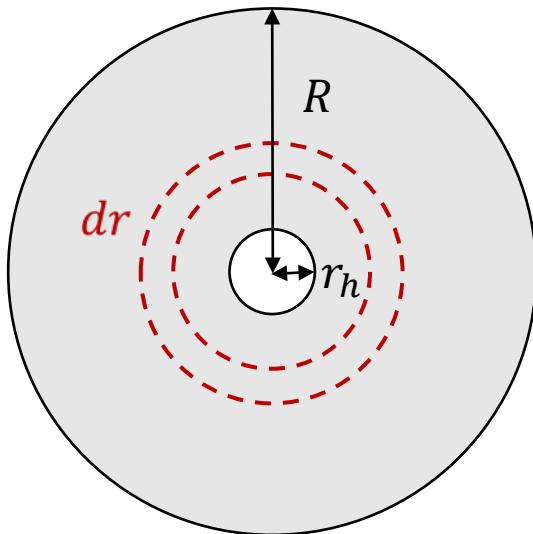
force reaching terminal depth

Contributes to upward force



Complexity of combined jet forces requires empirical modelling

Empirical Force Field



Consider upwards forces applied to disk
by jet as a non-uniform pressure field
over area

$$P(\mathbf{R}, \mathbf{v}_j, \mathbf{r}_j)$$

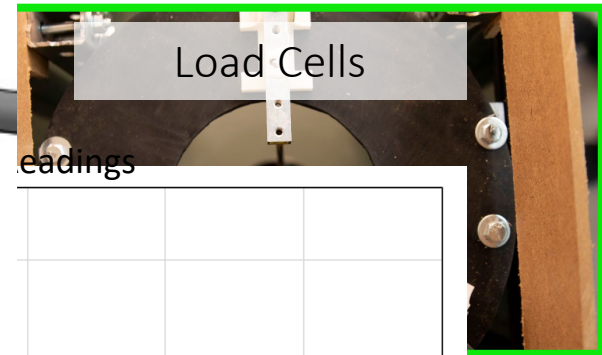
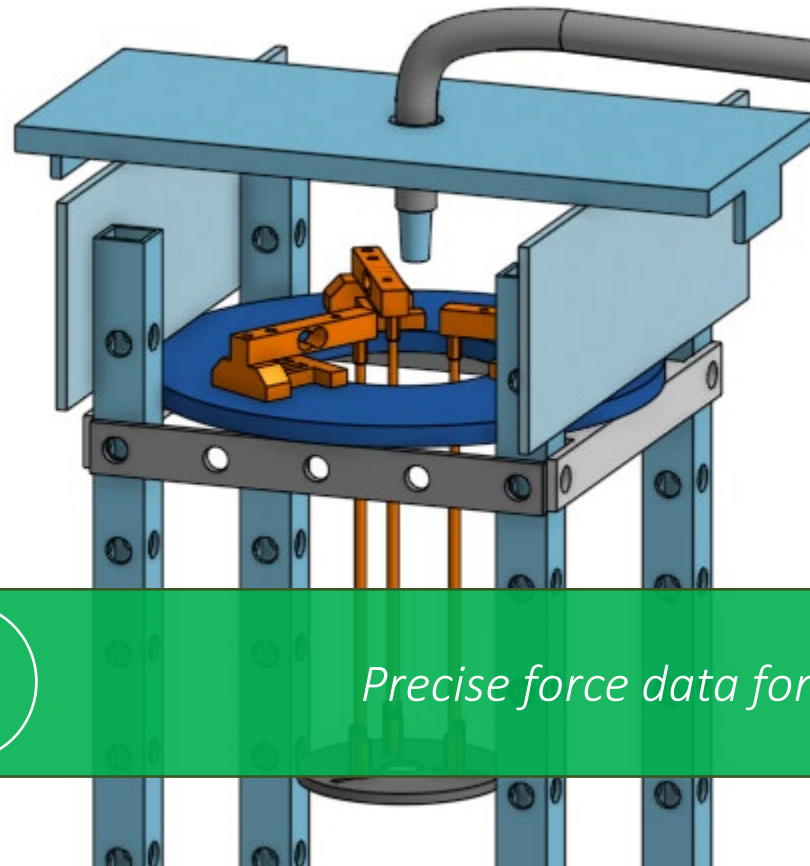
$$F_j = 2\pi \int_{r_h}^R P(\mathbf{R}, \mathbf{v}_j, \mathbf{r}_j) r dr$$

Area integral yields overall
force acting on disk

Q_j

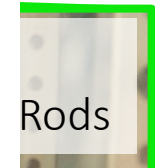
Experimentally isolate
each parameter to
visualize pressure
field

Force Measurement System



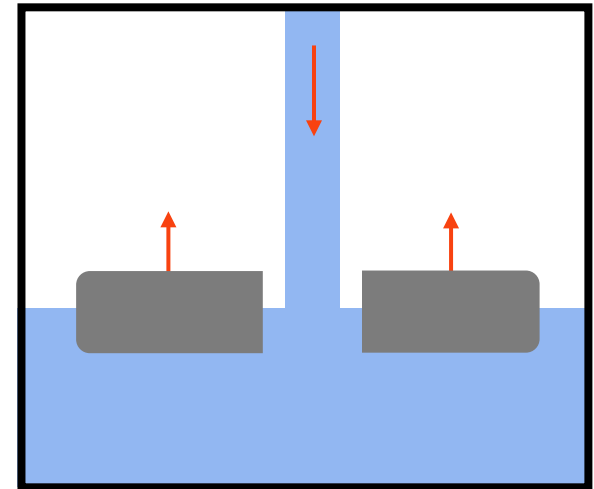
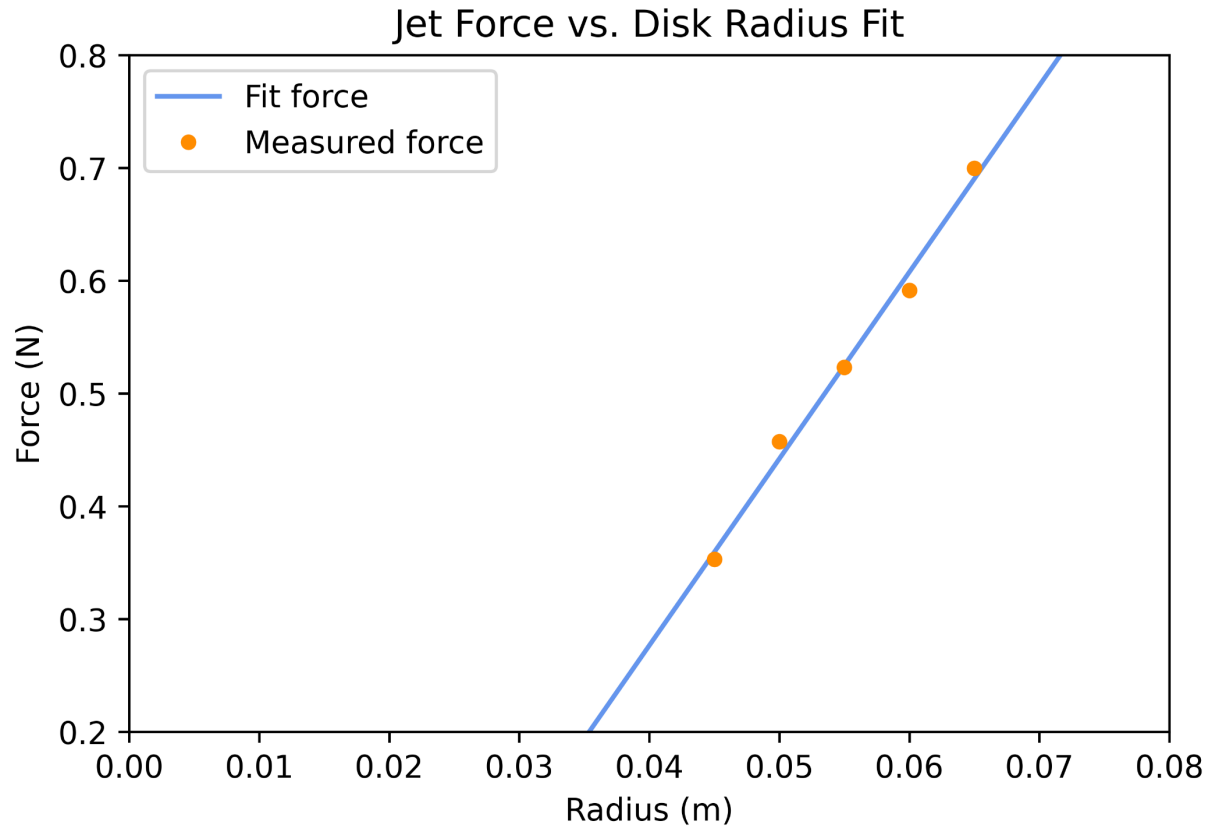
$$F = \alpha_1 F_1 + \alpha_2 F_2 + \alpha_3 F_3$$

23x - 1.0547
0.9999



Precise force data for empirical model

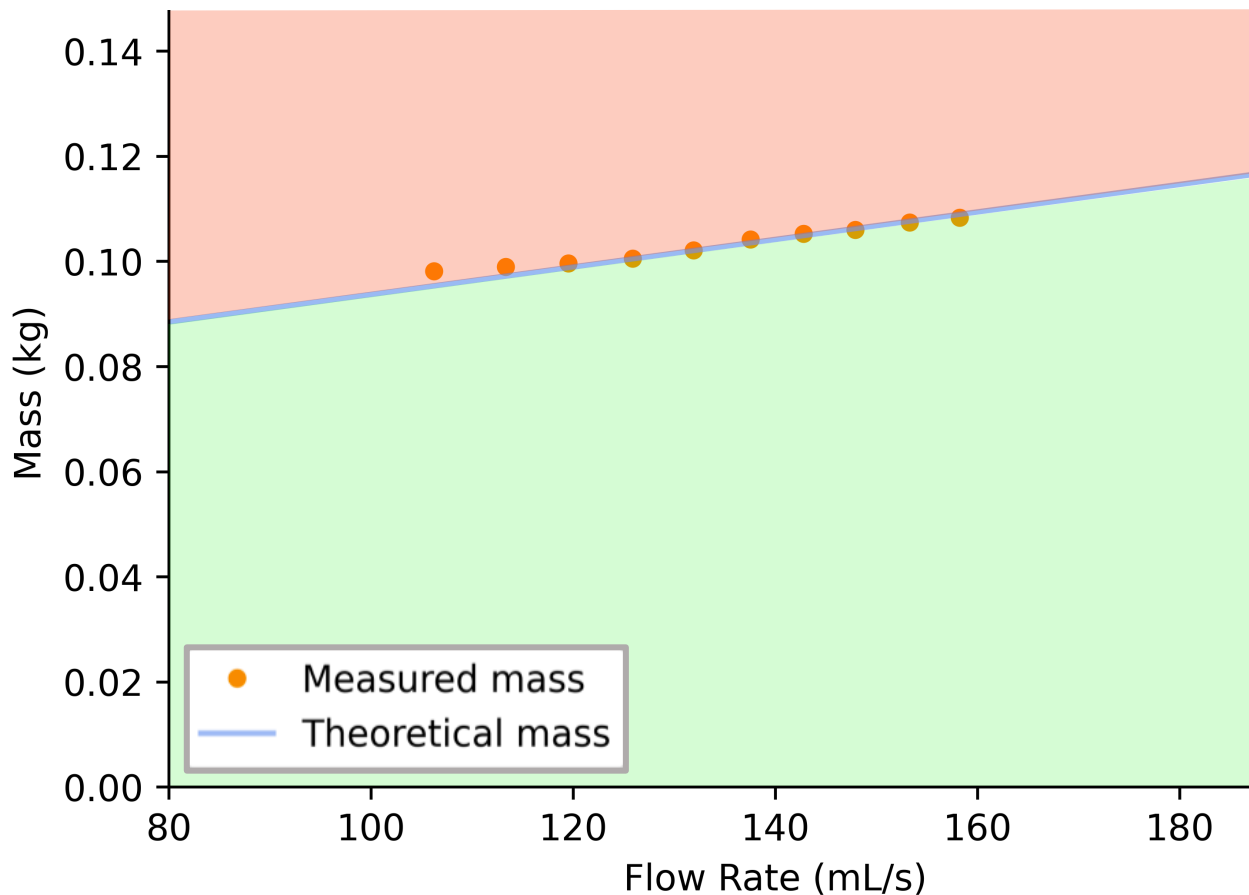
Empirical Force Fit



Flow Dynamics

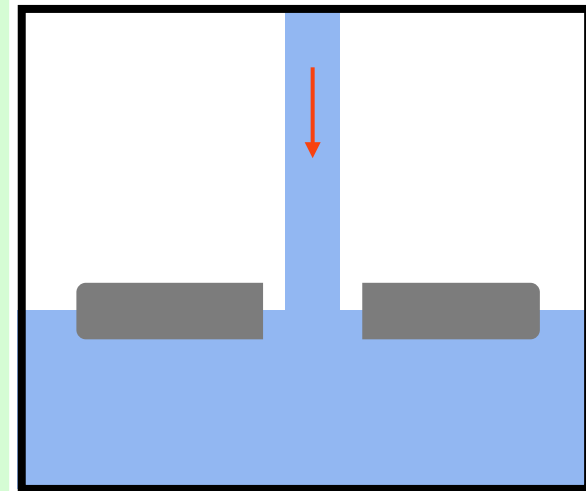
Hydraulic Jump

Experimental Verification



Equilibrium Equation

$$F_g - F_b - F_t - F_j = 0$$



Flow Dynamics

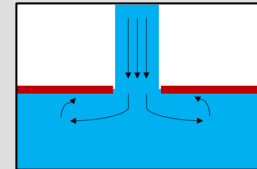
Hydraulic Jump

Theoretical Model

A

Flow Dynamics

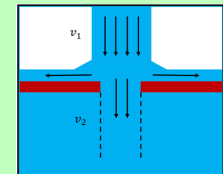
Force Analysis, Jet Effects, Empirical Model



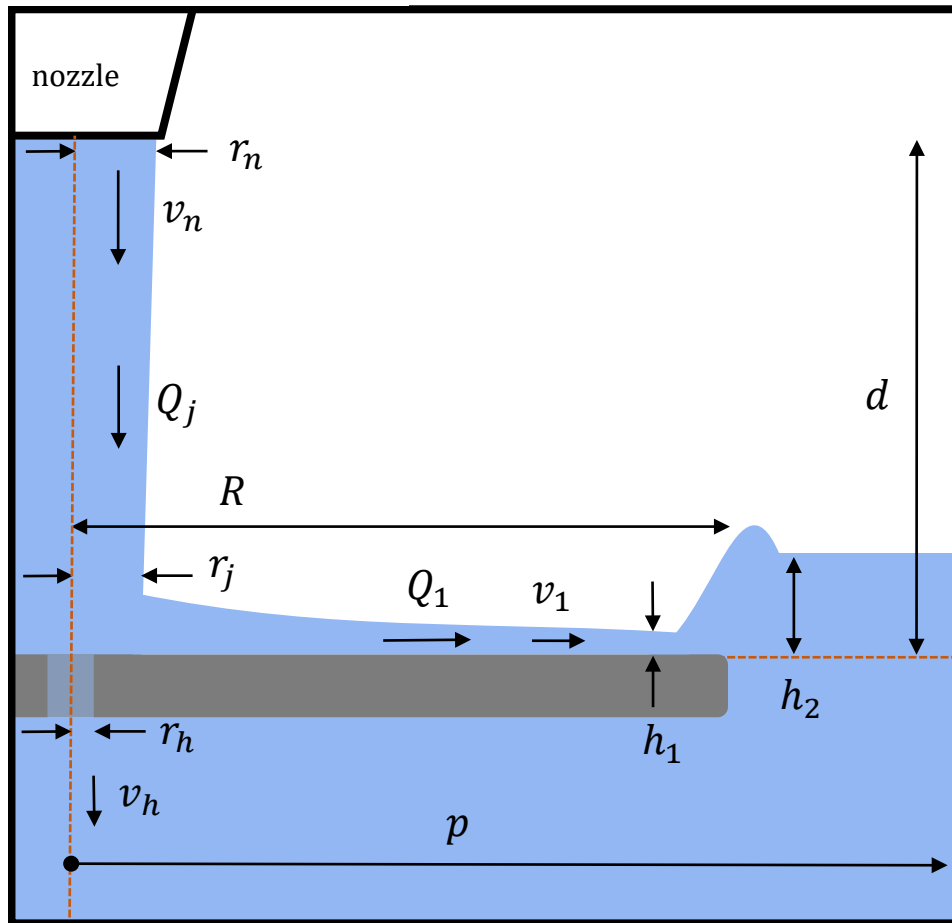
B

Hydraulic Jump

Impinging Force, Archimedes' Principle, Empirical Model



Geometry



Terms:

Q = volume flow rate

\dot{m} = mass flow rate

r = radius

v = fluid velocity

h = fluid height

d = nozzle height

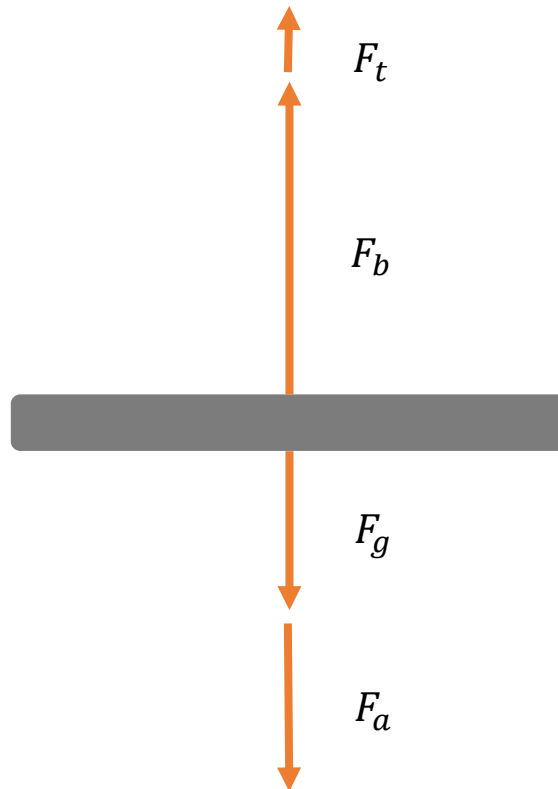
p = radial coordinate

m = mass

Flow Dynamics

Hydraulic Jump

Free Body Diagram



$F_g = \text{Force of gravity}$

$F_a = \text{Impinging force of jet}$

$F_b = \text{Buoyant force}$

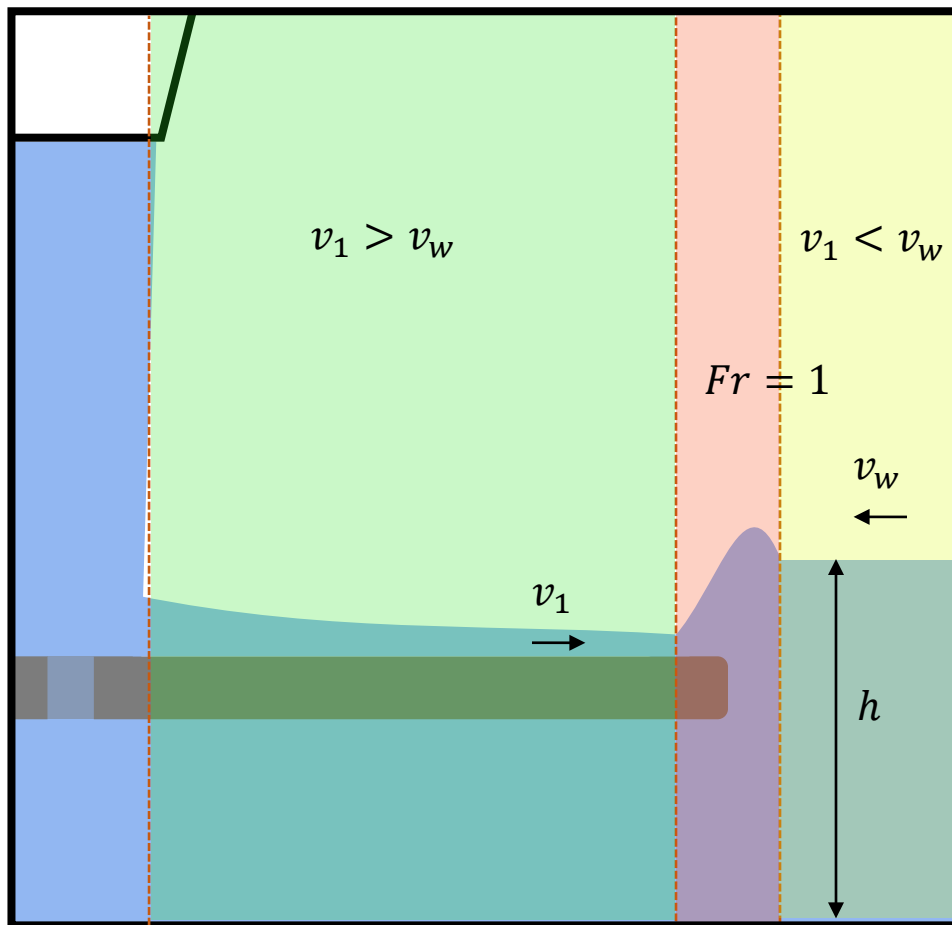
$F_t = \text{Force of surface tension}$

Equilibrium necessary to float:

$$F_g + F_a - F_b - F_t = 0$$

Hydraulic Jump

$$v_w = \sqrt{gh}$$



- Supercritical Region*
- Transition Region*
- Subcritical Region*

The Hydraulic Jump occurs in the transition region.

Depth (h) will rapidly increase and the jump will occur just beyond radius of disk

Flow Dynamics

Hydraulic Jump

Additional Force

Terms:

$F_a = \text{impinging force}$

$F_w = \text{force of gravity of water}$

New force equilibrium equation:

$$2\pi R\gamma \cos(\theta) + \rho gV = mg + F_w + F_a$$

Surface
Tension

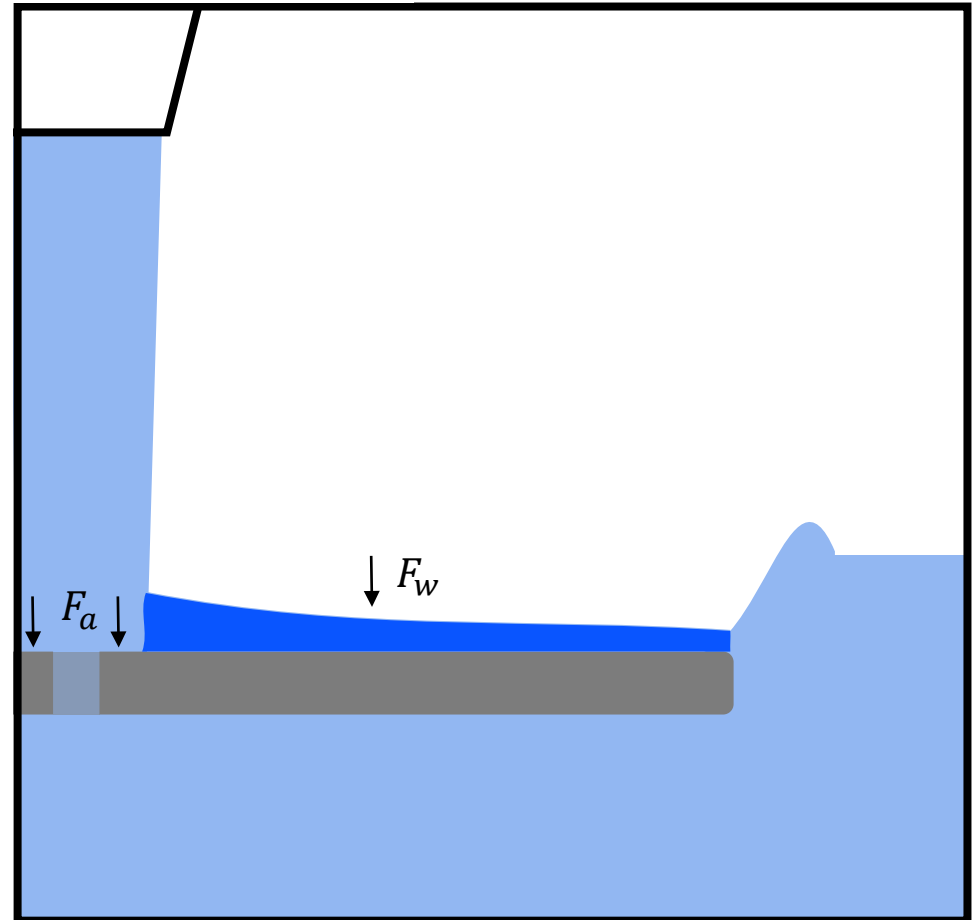
Buoyant Force

Water Weight

Weight

Impinging
Force

Solve For F_w , F_a , and V



Narrowing Jet

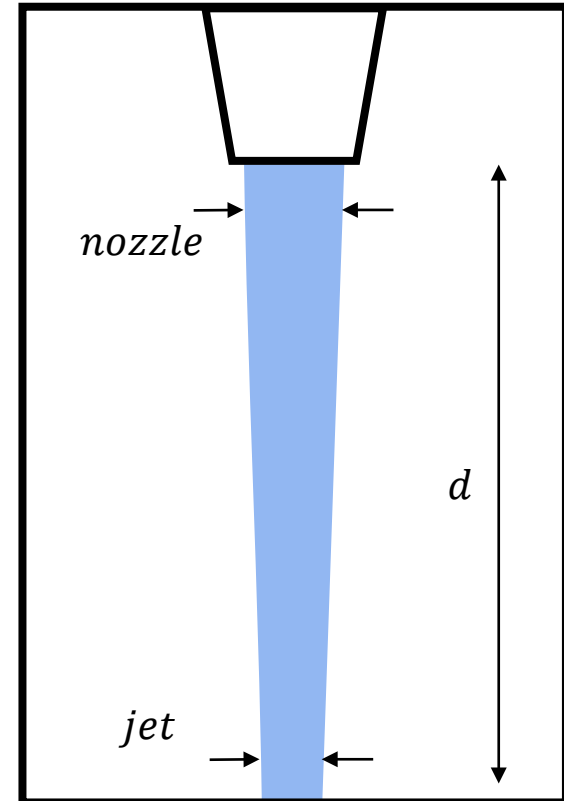
The dimensions of the jet will change after accelerating for distance d

Use Bernoulli's Principle to find velocity:

$$v_j = \sqrt{v_n^2 + 2gd}$$

Use Continuity to find radius:

$$r_j = \sqrt{\frac{Q_j}{v_j \pi}}$$



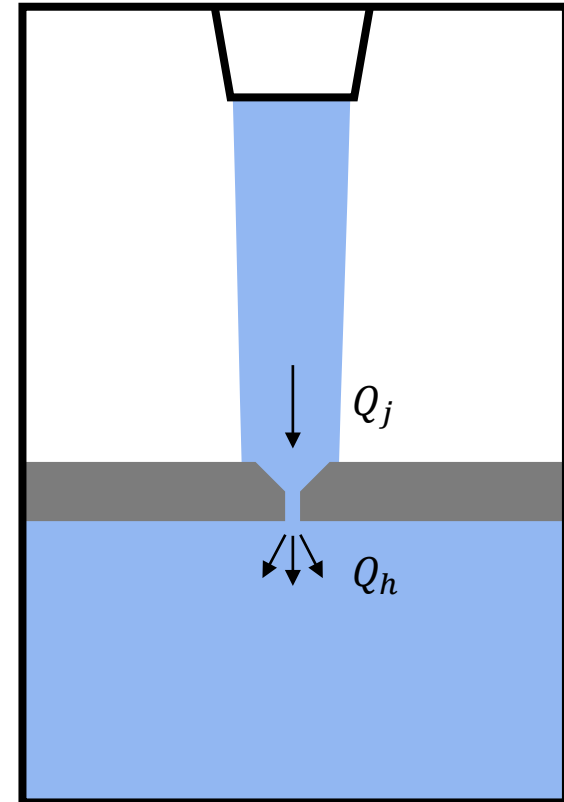
Hole Flow Rate

In order for the system to be continuous, the flow rate escaping through the hole must be found.

Due to the indent in the disk for stability, this is very hard to model.

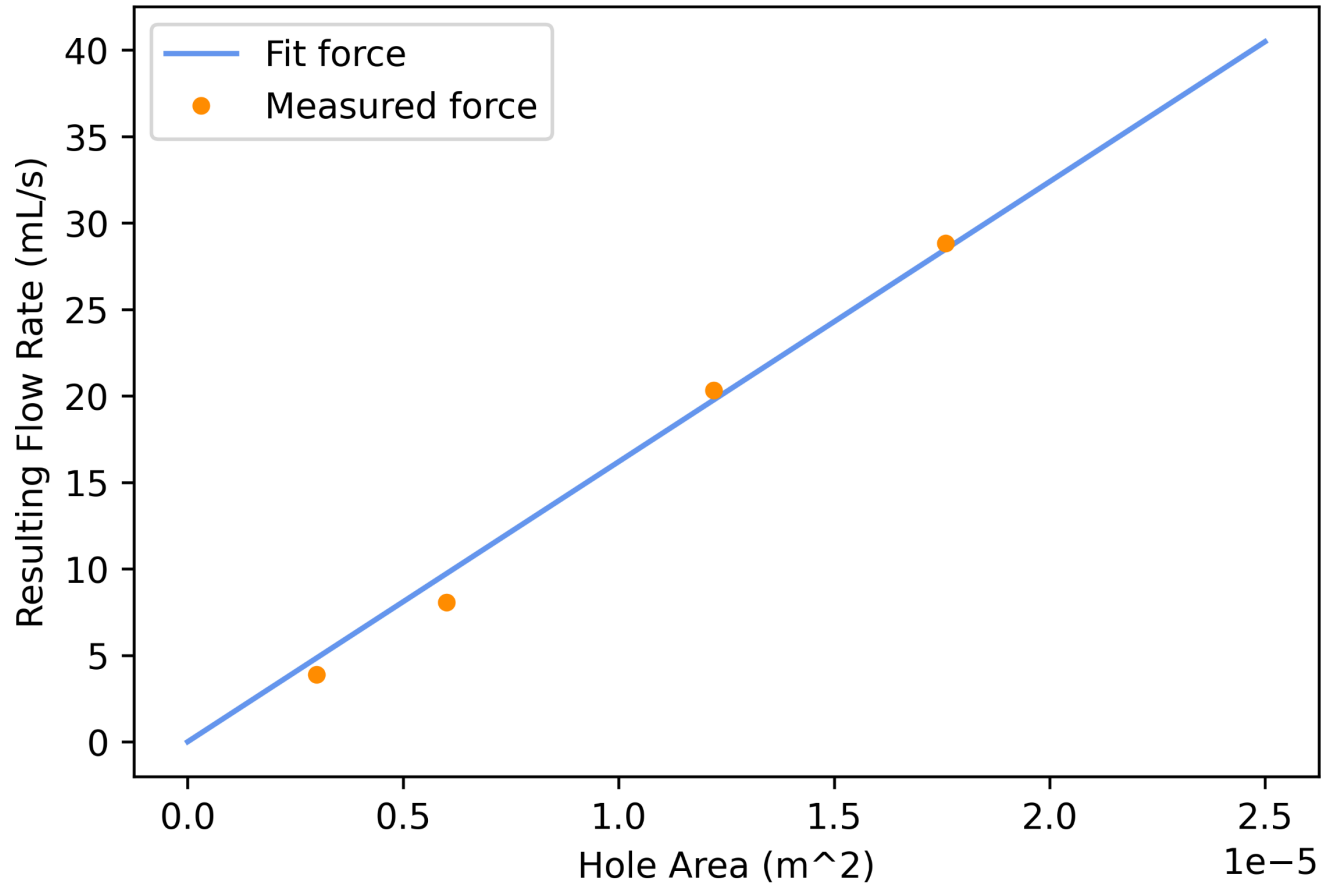
We will rely on an empirical fit

$$Q_h(Q_j, \frac{r_h}{r_j})$$



Hole Flow Rate

Flow Rate Through the Hole vs. Hole Area



Flow Dynamics

Hydraulic Jump

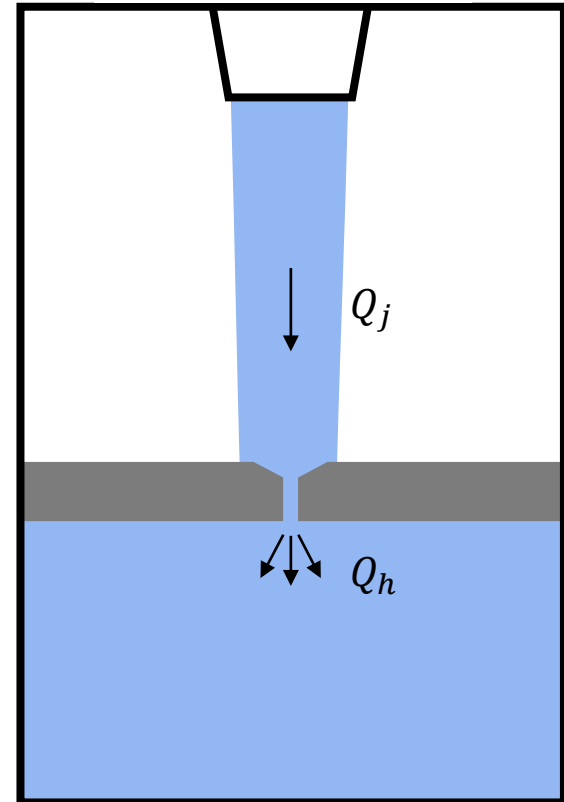
Impinging Force

To find the force of the jet hitting the plate, we can solve for linear momentum. Using the previously fitted mass flow rate through the hole

$$F_a = \dot{m}_j v_j - \dot{m}_h v_h$$

$$F_a = v_j^2 \rho A_j - v_h^2 \rho A_h$$

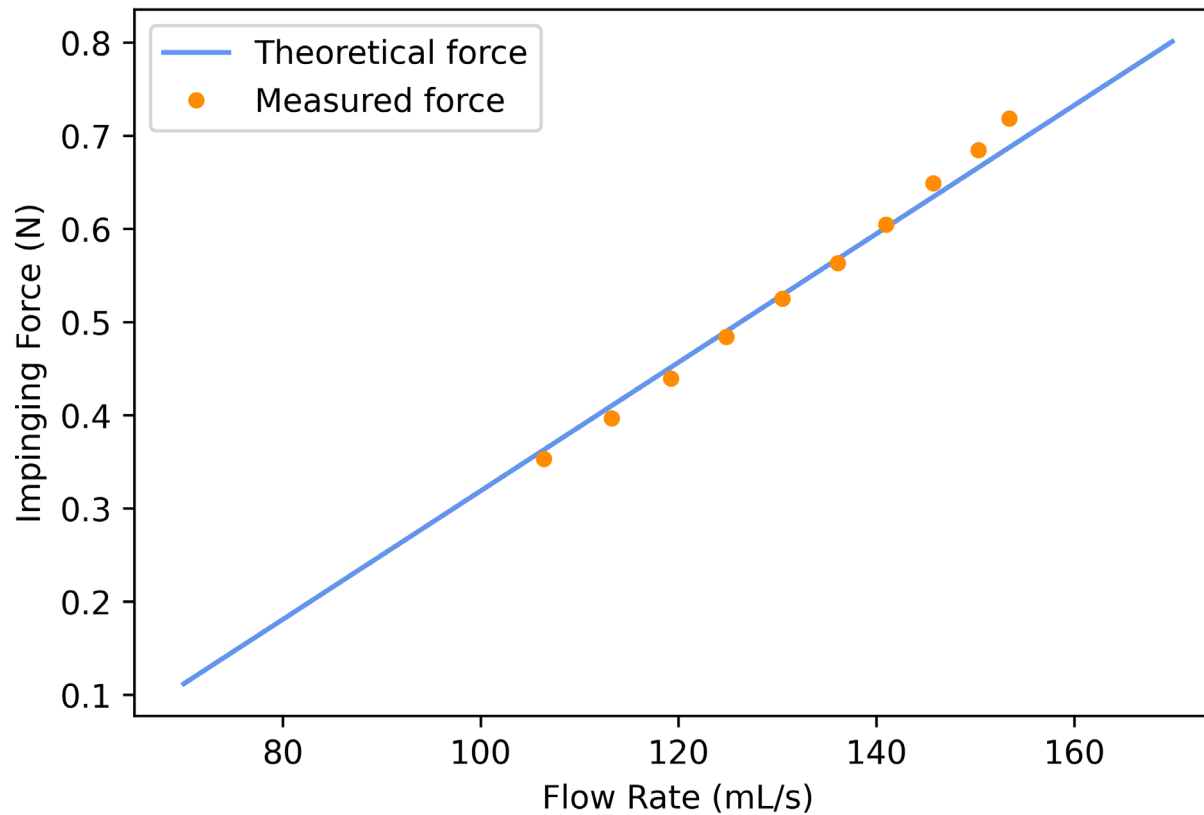
$$F_a = \rho(Q_j v_j - Q_h v_h)$$



Impinging Force Experimental

Using Force Measurement System

Impinging Force vs. Total Flow Rate



Flow Dynamics

Hydraulic Jump

Water Weight

Similar to the Jet Force Field, we can find h_1 as a function of radial coordinate p by integrating the area integrals to find water volume.

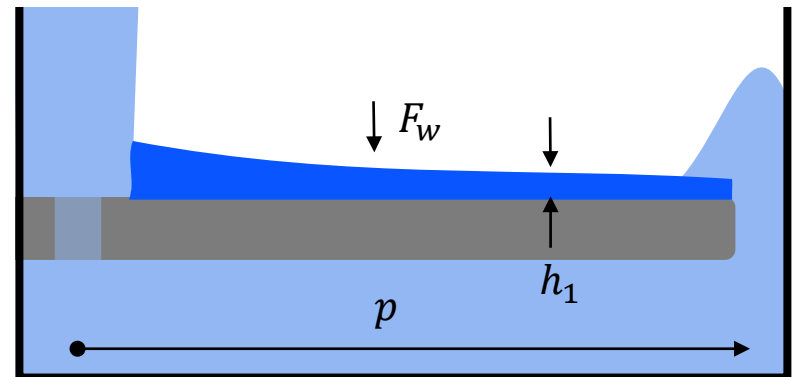
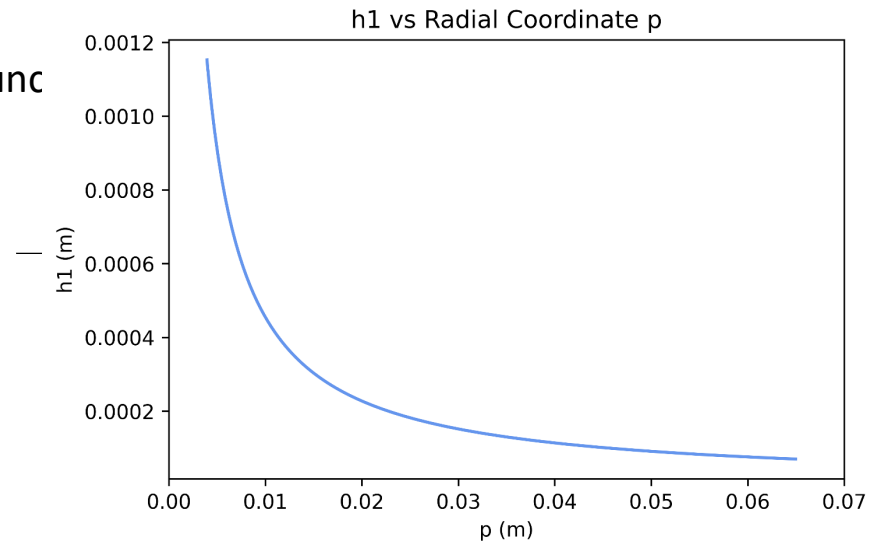
Assuming the Hydraulic Jump Radius is larger than the disk radius, we can derive an equation for height from energy conservation and continuity:

$$h_1 = \frac{(Q_j - Q_h)^{\frac{3}{2}}}{p \cdot 2\pi \sqrt{Q_j(v_n^2 + 2gd) - Q_h v_h^2}}$$

$$V = 2\pi \int_{r_h}^R h_1(p) dp = \frac{(Q_j - Q_h)^{\frac{3}{2}}}{\sqrt{Q_j(v_n^2 + 2gd) - Q_h v_h^2}} (R - r_h)$$

$$F_w = \rho g V$$

Flow Dynamics

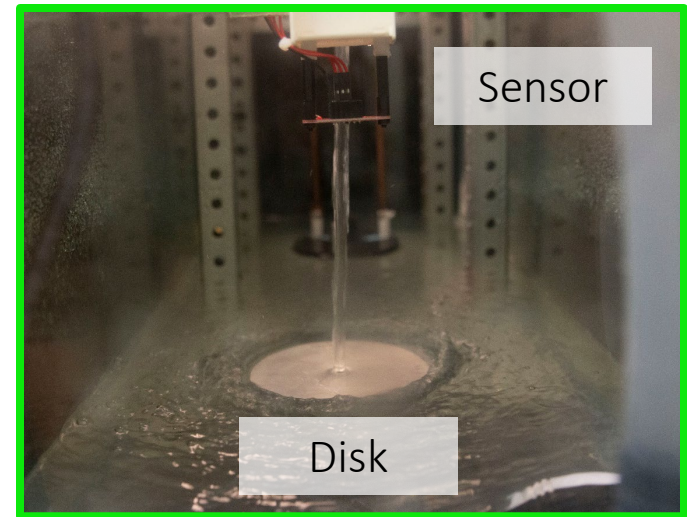
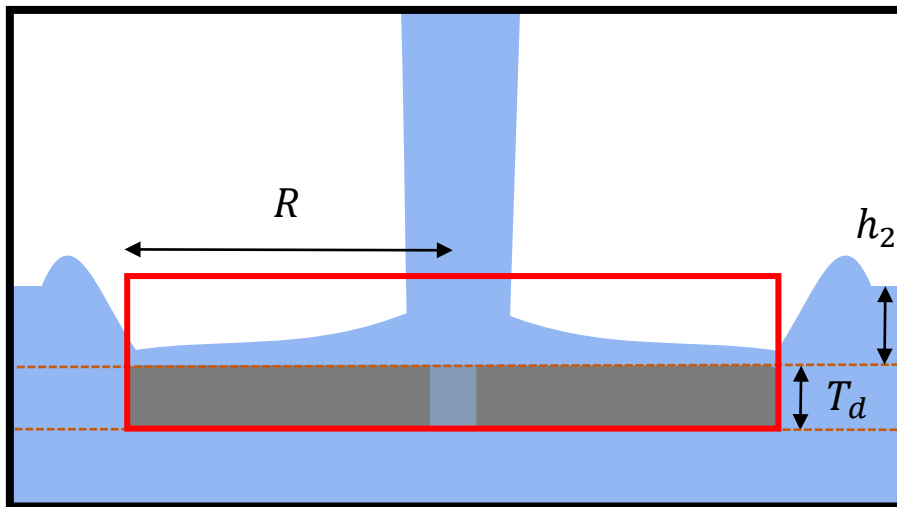


Hydraulic Jump

Buoyant Force - Range Sensor

Buoyant force \propto the depth of the disk
below the water surface (h_2).

$$F_b = \rho g V = \rho g (\pi R^2 - \pi r_h^2) (h_2 + T_d)$$



Specifications:

VL6180 Time-of-Flight Range Sensor

0.000m – 0.250m \pm 0.0005m

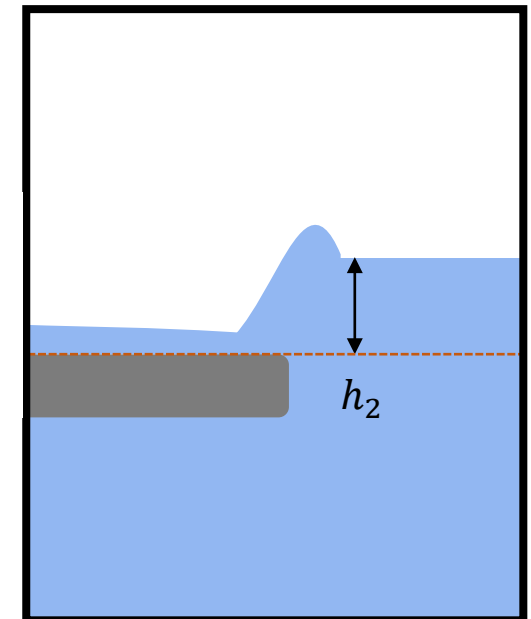
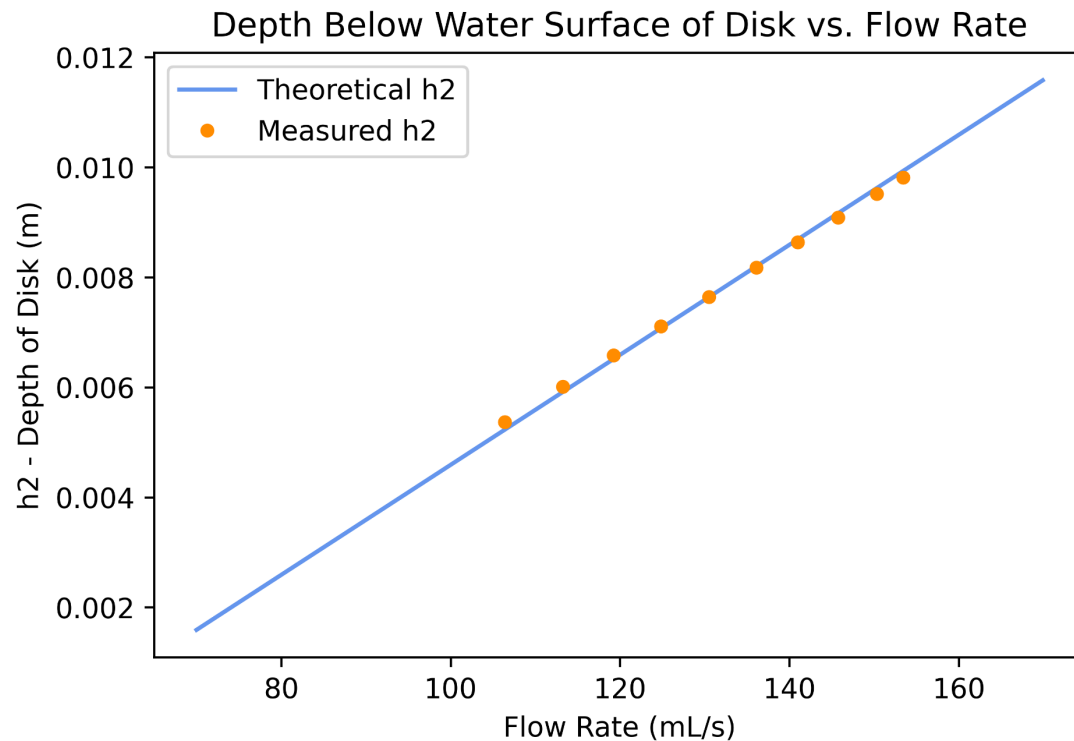
To measure change in height of the disk

Height Solution

Applying the equilibrium equation, we predict a value h_2 and compare to the Range Sensor Data

Equilibrium Equation

$$2\pi R\gamma \cos(\theta) + \rho gV = mg + F_w + F_a$$

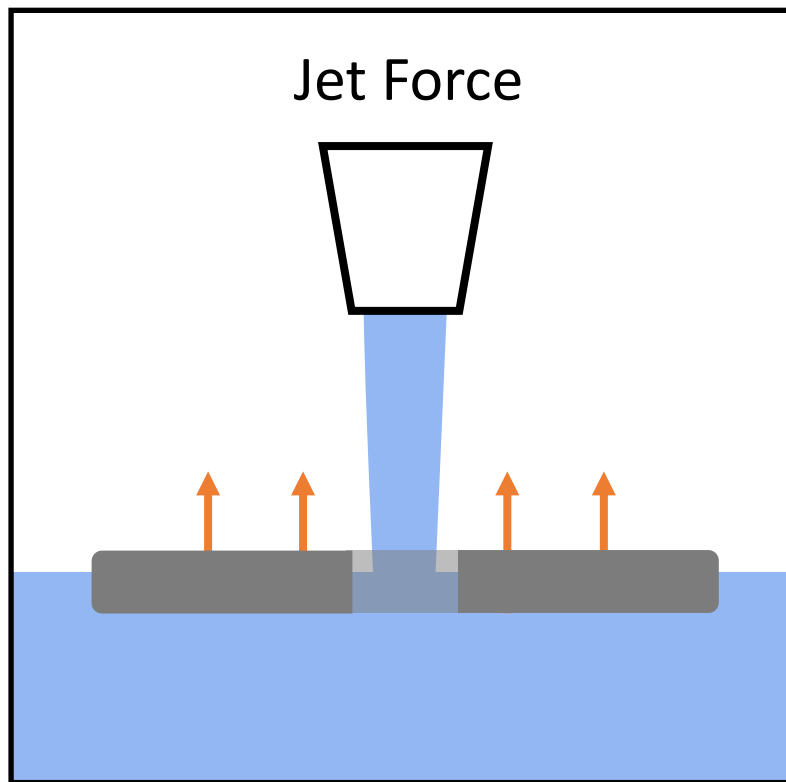


Flow Dynamics

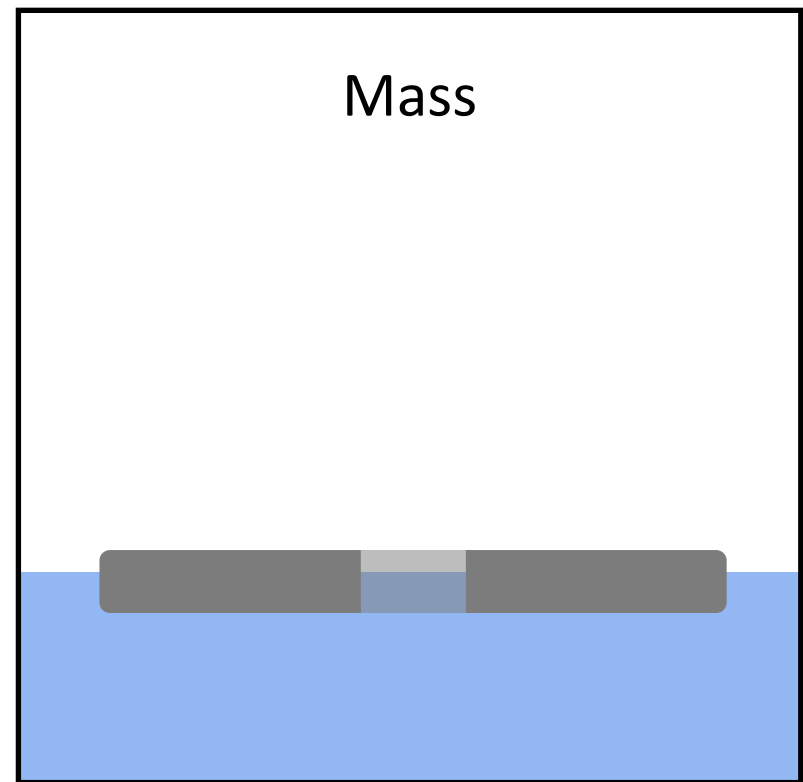
Hydraulic Jump

Key Parameters

Key Parameters – Flow Dynamics

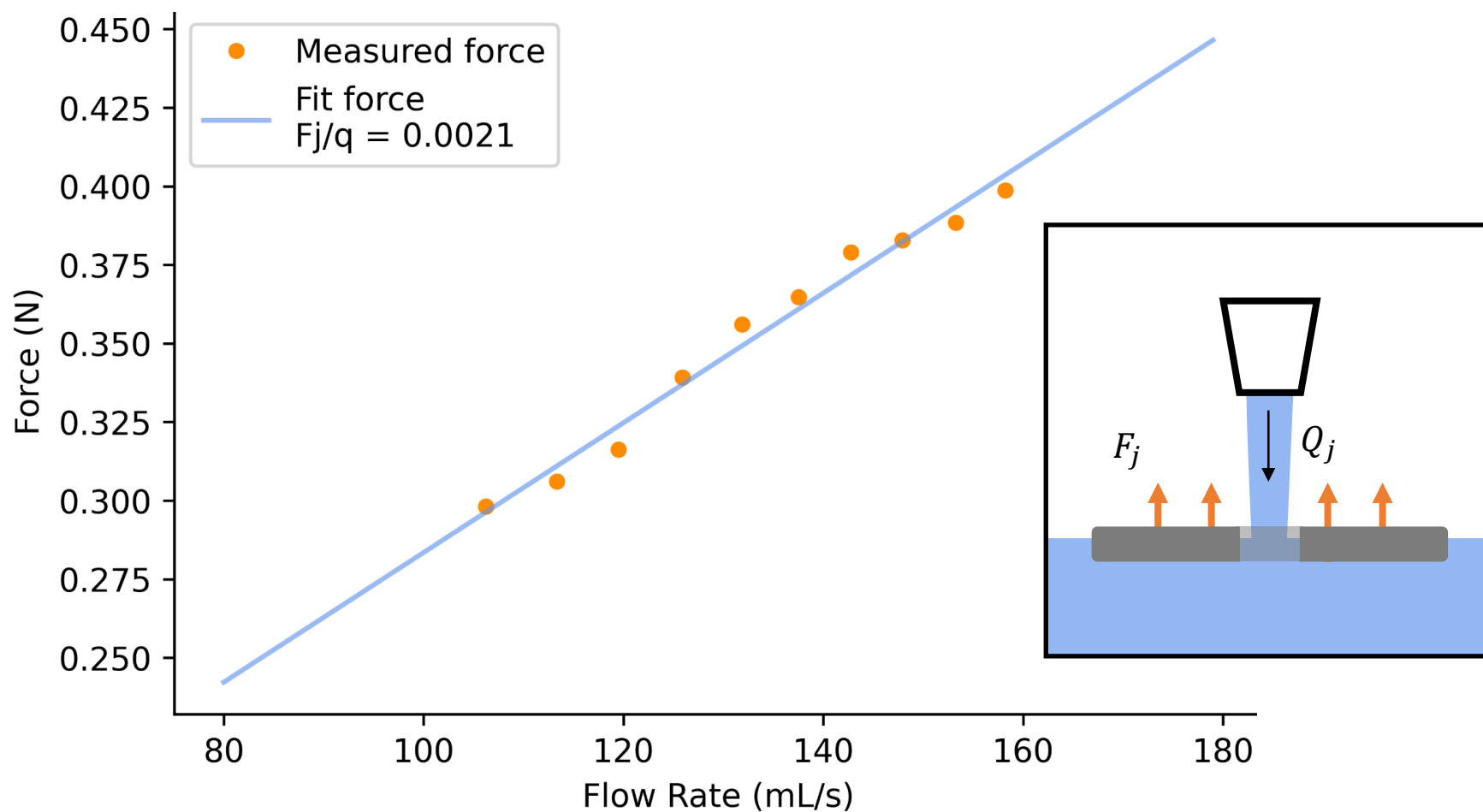


Float

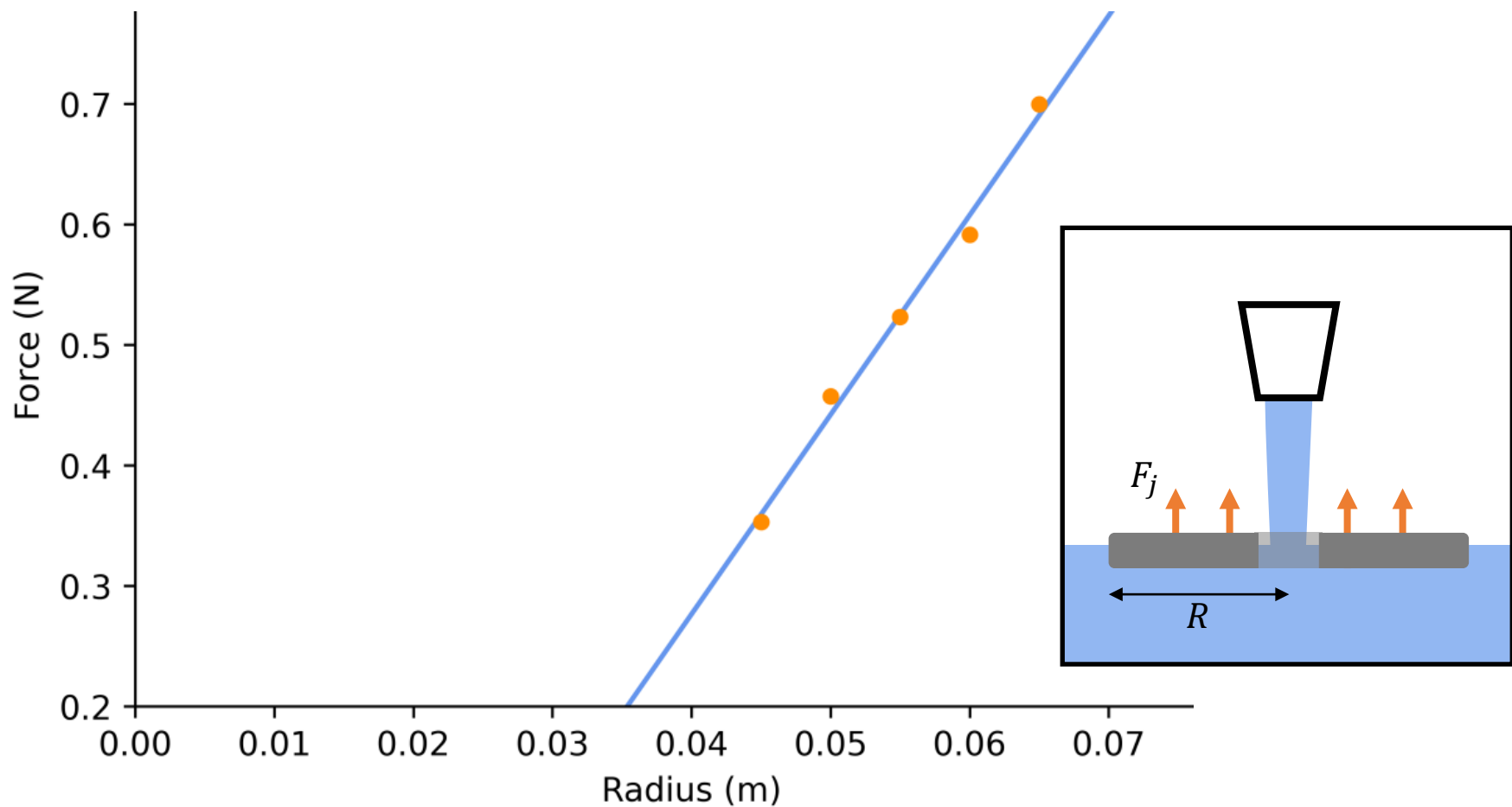


Sink

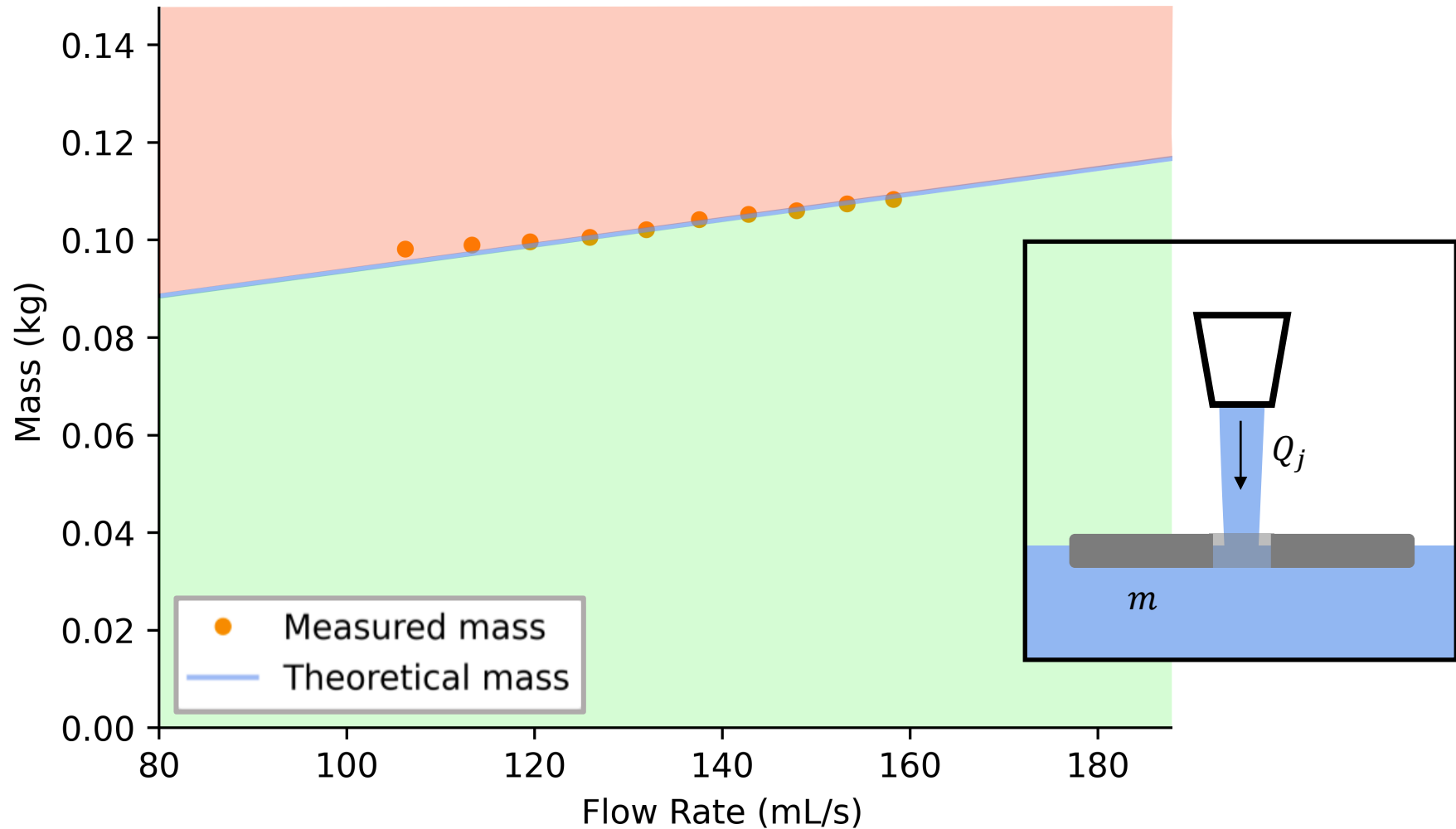
Jet Force vs. Flow Rate



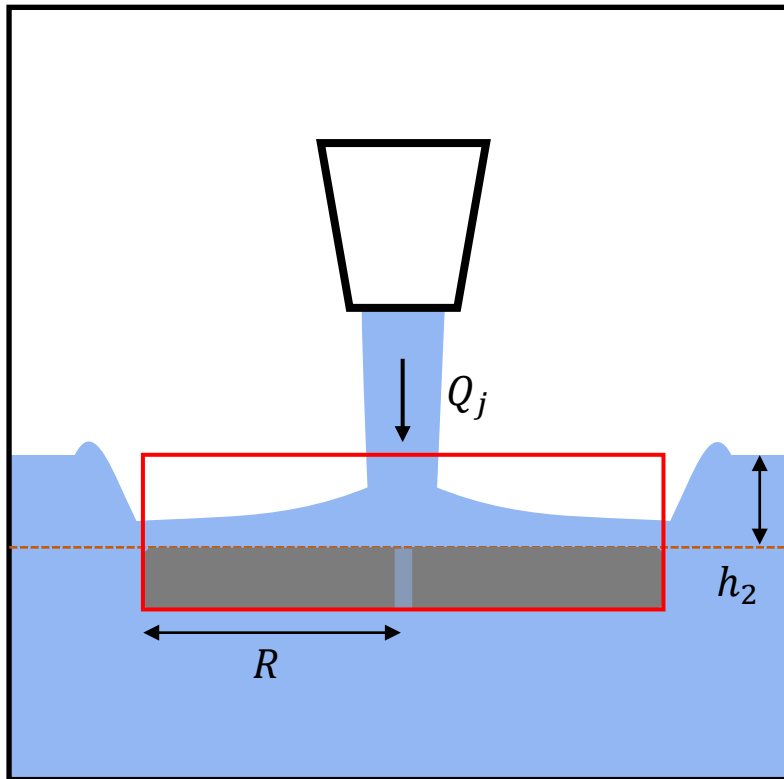
Jet Force vs. Disk Radius



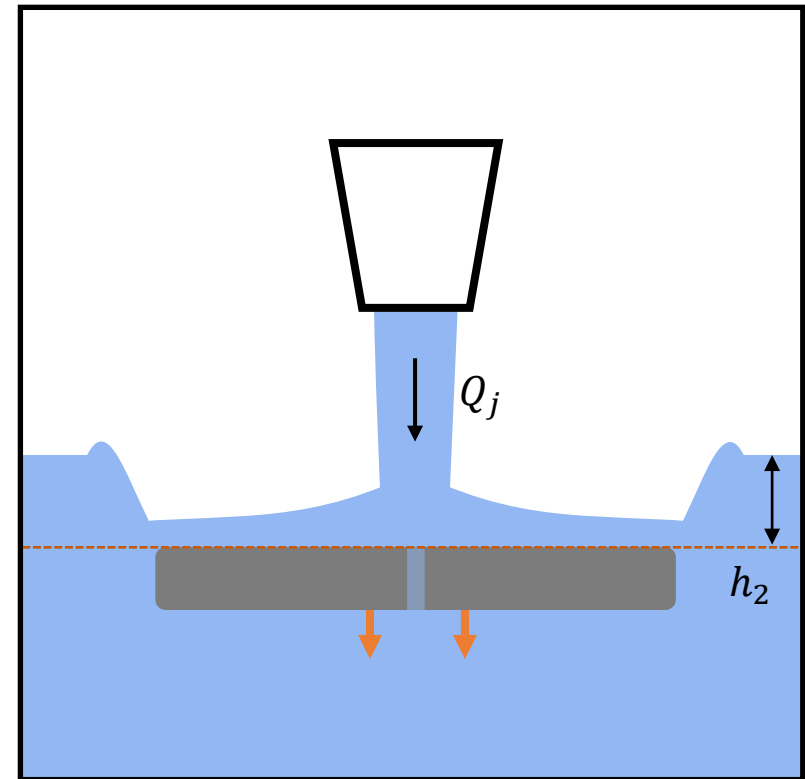
Maximum Mass vs. Flow Rate



Key Parameters – Hydraulic Jump

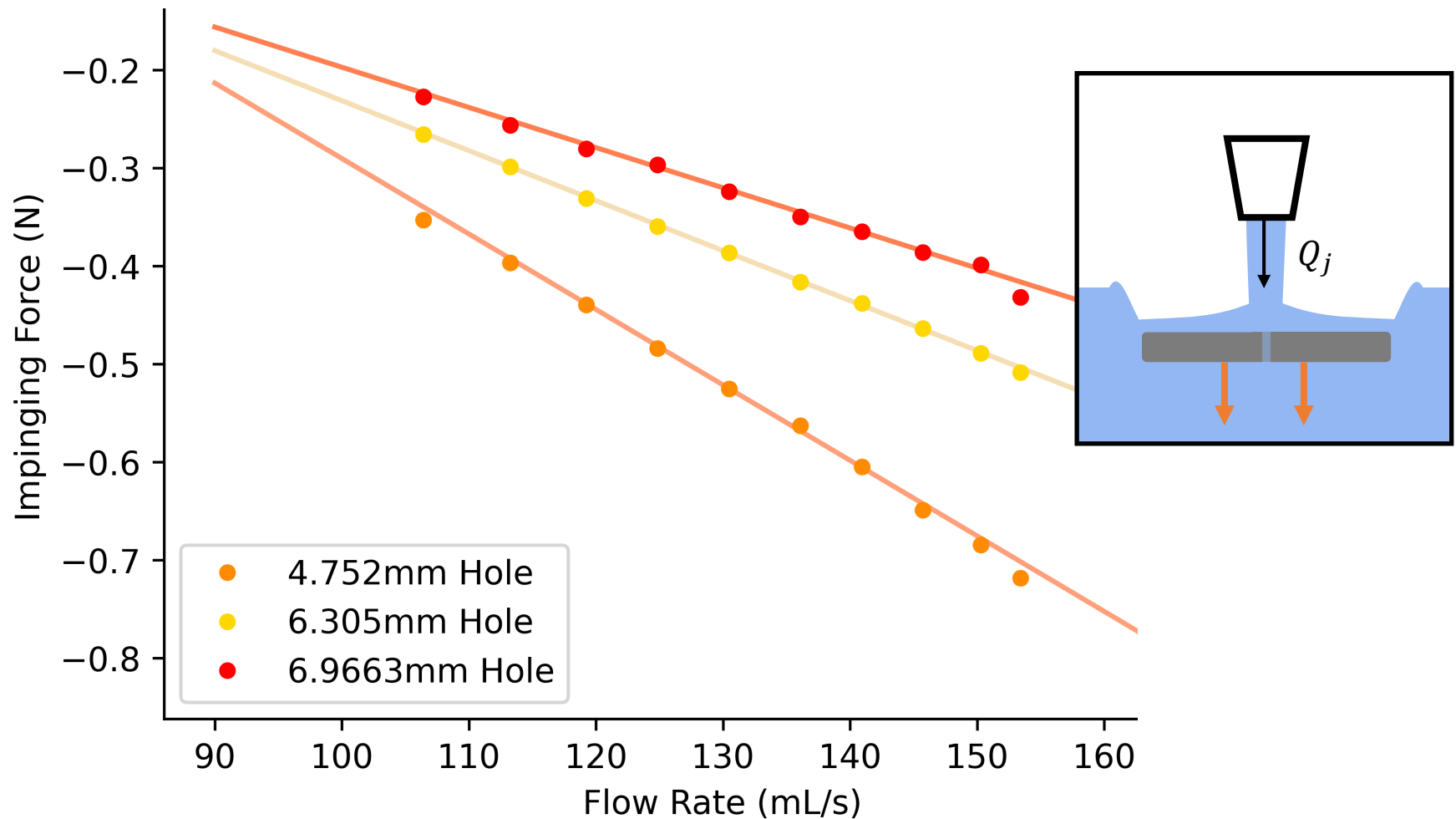


Buoyant Force

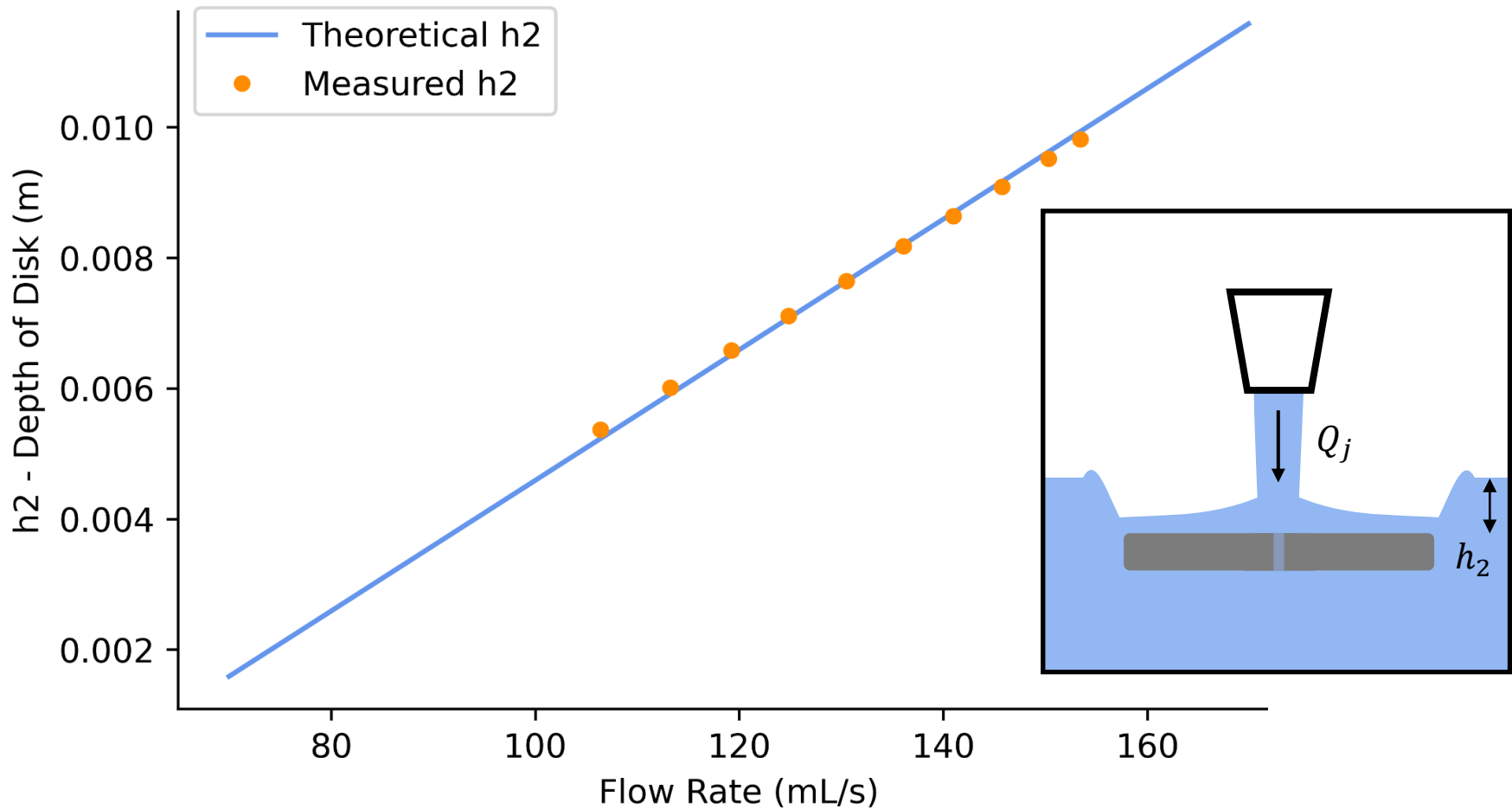


Impinging Force

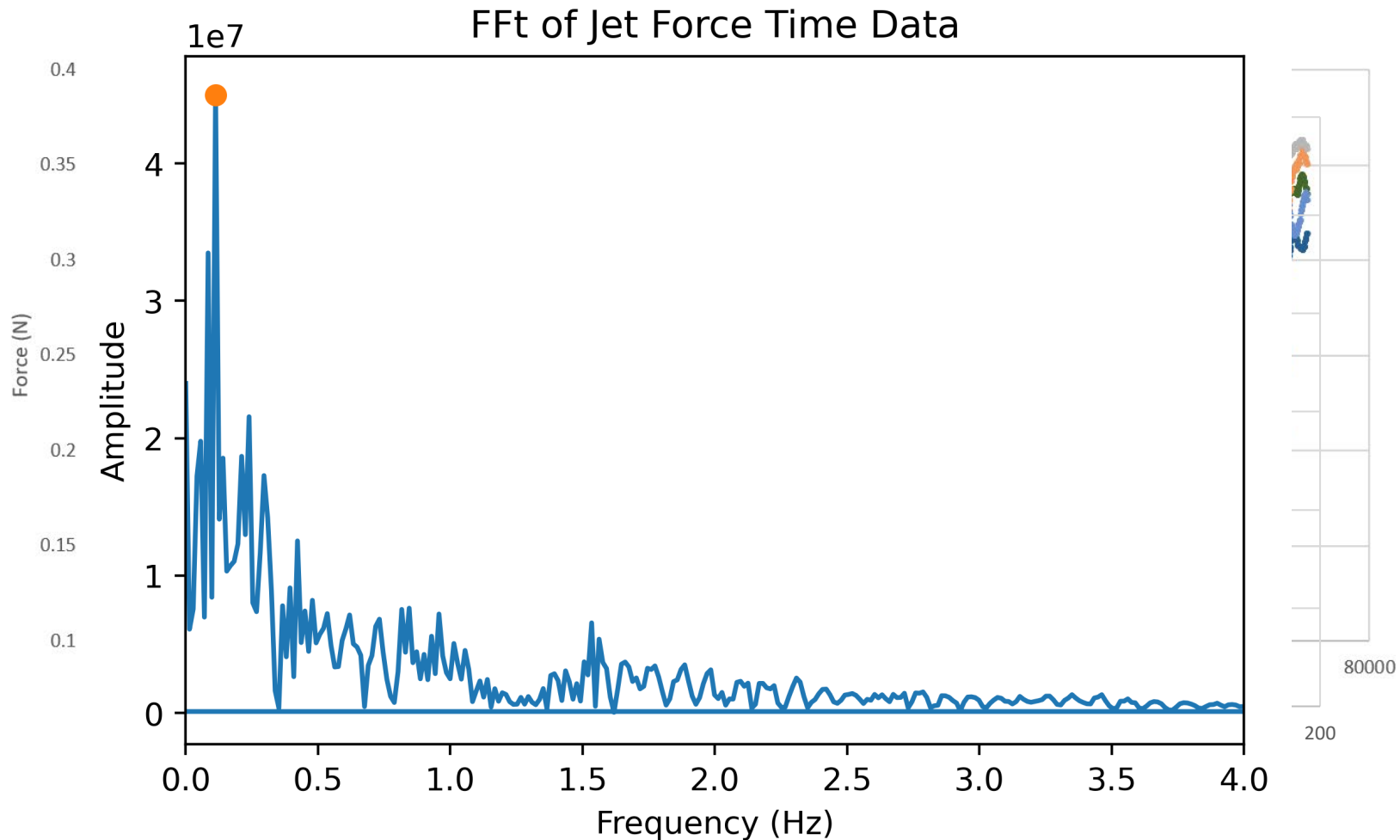
Impinging Force vs Flow and Hole



Disk Depth (buoyancy) vs. Flow Rate



Further Insights



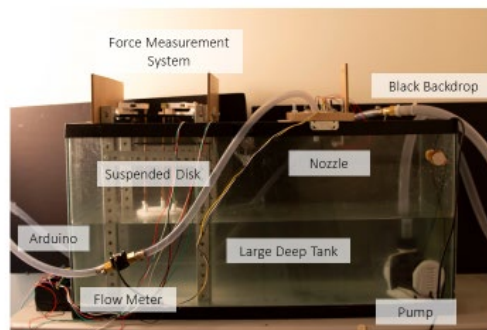
Conclusion

*“A metal disk with a **hole** at its centre **sinks** in a container filled with **water**. When a **vertical water jet** hits the **centre of the disc**, it may **float** on the water surface. **Explain this phenomenon and investigate the relevant parameters.**”*

Controlled experimental setup

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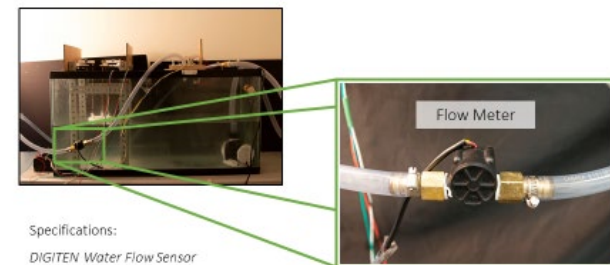
Experimental Setup



Introduction Experimental Setup Theoretical Model Key Parameters Conclusion

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Flow Meter



Specifications:
DIGITEN Water Flow Sensor
1L/min – 30L/min
Pulse Counter with Arduino

Introduction Experimental Setup Theoretical Model Key Parameters Conclusion

Introduction

Experimental Setup

Theoretical Model

Key Parameters

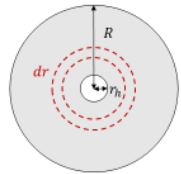
Conclusion

Conclusion

*“A metal disk with a **hole** at its centre **sinks** in a container filled with **water**. When a **vertical water jet** hits the **centre of the disc**, it may **float** on the water surface. **Explain this phenomenon and investigate the relevant parameters.**”*

Thorough theoretical model for both cases

Empirical Force Field



Consider upwards forces applied to disk by jet as varying pressure field over area

$$F_j = 2\pi \int_{r_h}^R P(R, v_j, r_j) dr$$

Area integral yields overall force acting on disk

Experimentally isolate each parameter to visualize pressure field

Flow Dynamics Hydraulic Jump

Water Weight

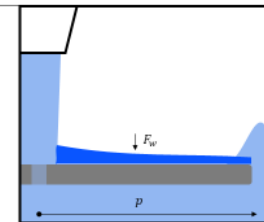
Similar to the Jet Force Field, we can find h_1 as a function of radial coordinate p and take area integrals to find water volume.

Assuming the Hydraulic Jump Radius is larger than the disk Radius, from energy conservation and continuity, we derive an equation for height:

$$h_1 = \frac{(Q_j - Q_h)^{\frac{3}{2}}}{p \cdot 2\pi \sqrt{Q_j(v_n + 2gd) - Q_h v_n^2}}$$

$$V = 2\pi \int_{r_h}^R h_1(p) dp = \frac{(Q_j - Q_h)^{\frac{3}{2}}}{\sqrt{Q_j(v_n + 2gd) - Q_h v_n^2}} (R - r_h)$$

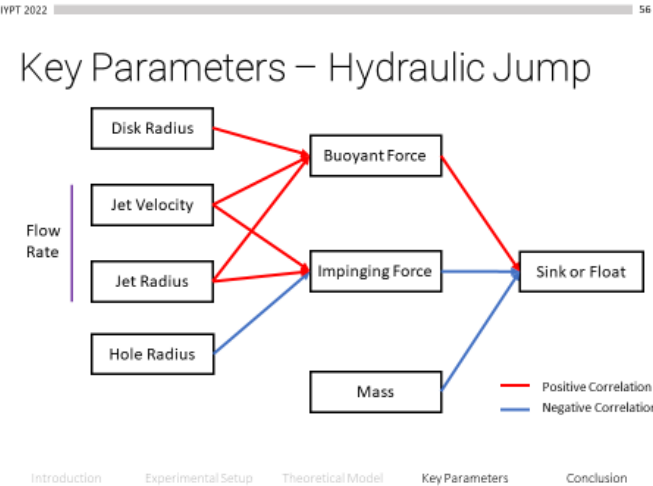
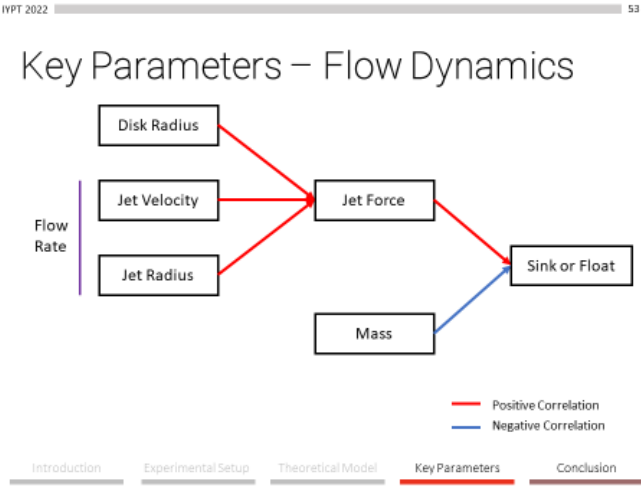
$$F_w = \rho g V$$



Conclusion

*“A metal disk with a **hole** at its centre **sinks** in a container filled with **water**. When a **vertical water jet** hits the **centre of the disc**, it may **float** on the water surface. **Explain this phenomenon and investigate the relevant parameters.**”*

Varied key parameters with experimental verification



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Thank you for listening

Appendix

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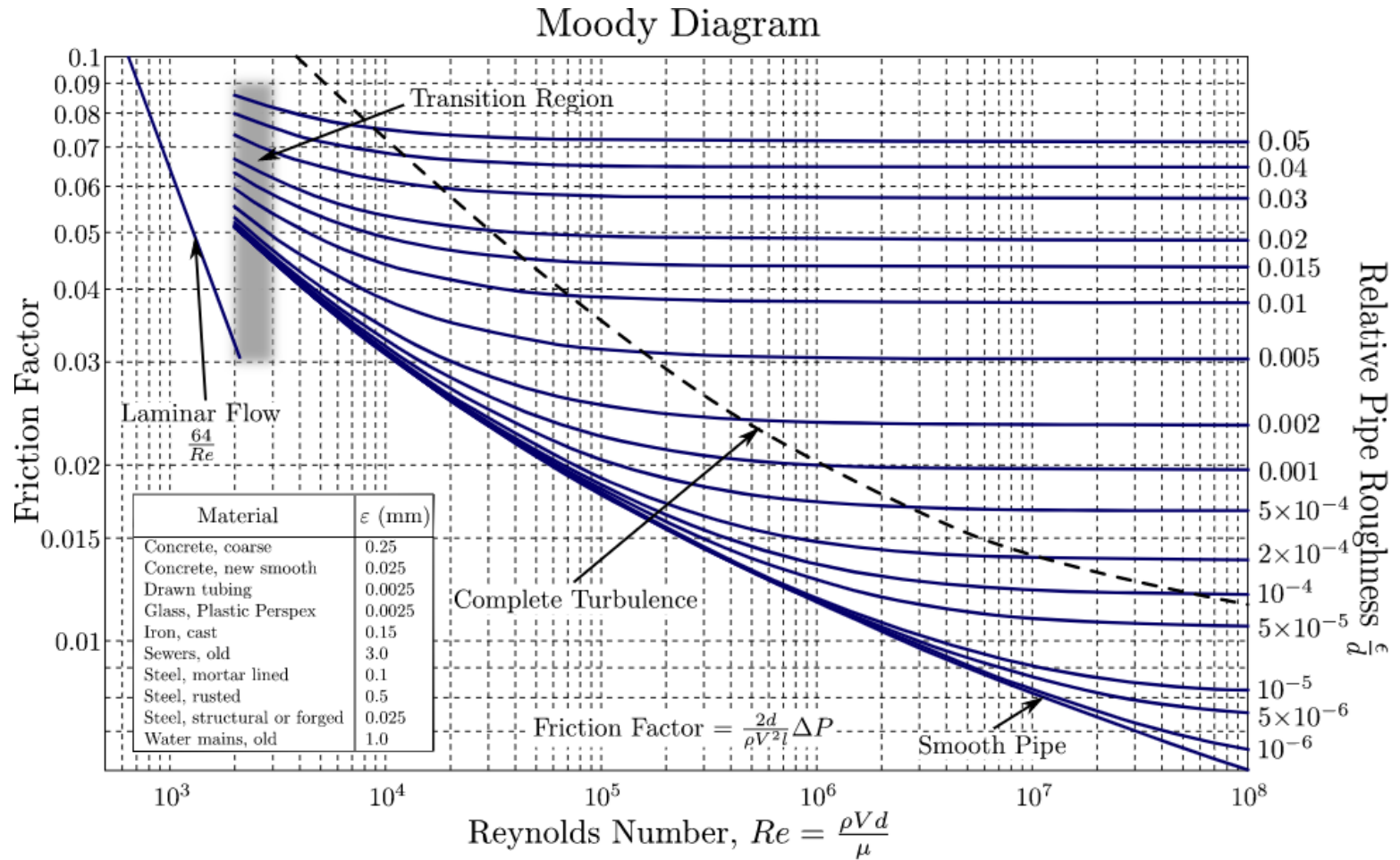
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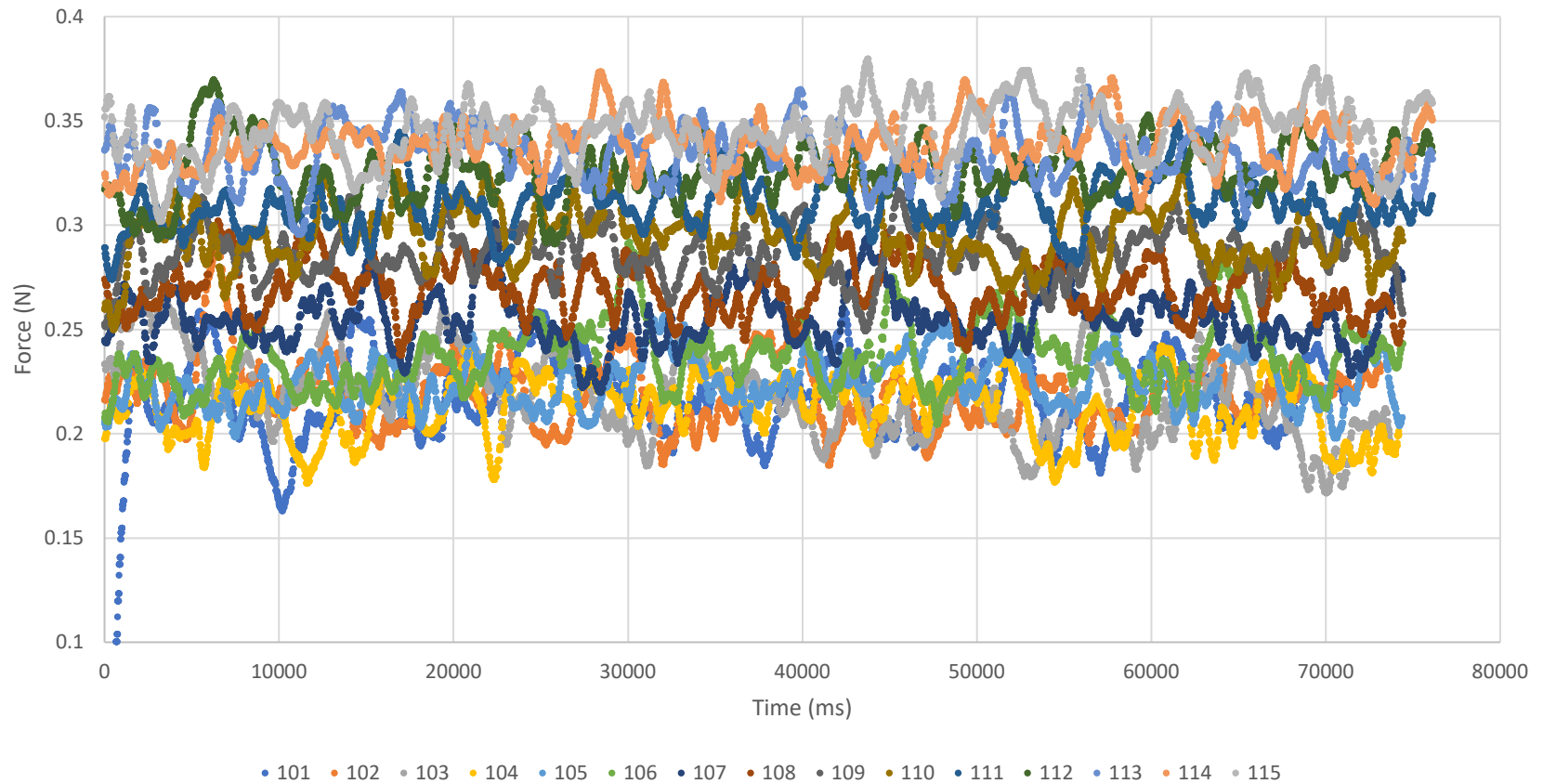
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Appendix A: Moody Diagram



Appendix B: Consistent Force Measurement

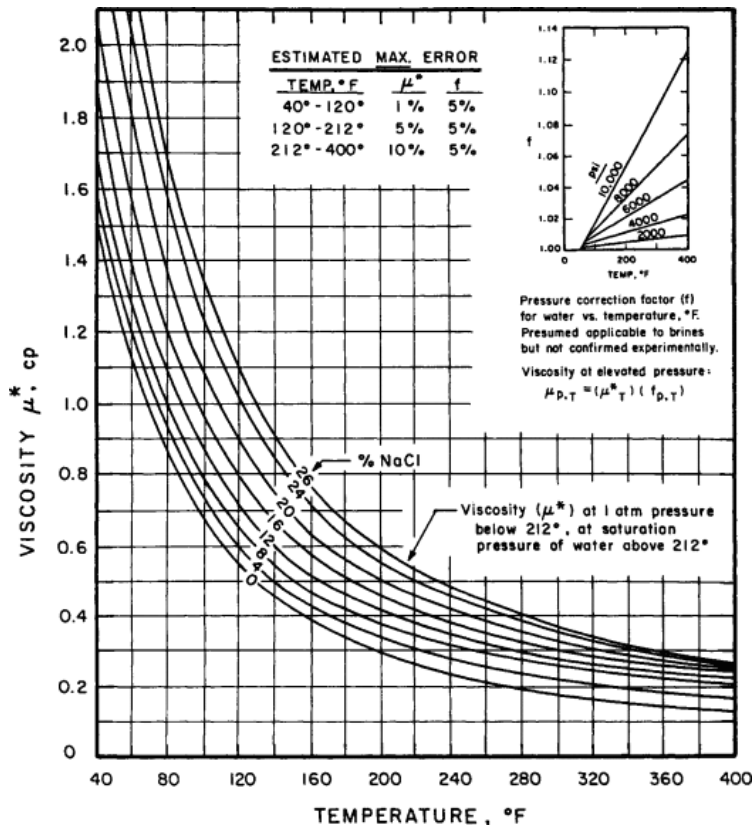
Jet Force with 15 Flow Rates vs. Time



Appendix C: Flow Meter Code

```
void loop() {
  interrupts(); //Enables interrupts on the Arduino
  if (restart == true){
    ini_time = micros();
    restart = false;
  }
  if (count >= 5){
    time_elapsed = micros() - ini_time;
    flowRate = 1000000 * count * slope / time_elapsed;
    Serial.print("flow rate: ");
    Serial.print(flowRate);
  }
  count = 0;
  restart = true;
}
```

Appendix D: Temperature and Density of Fluid

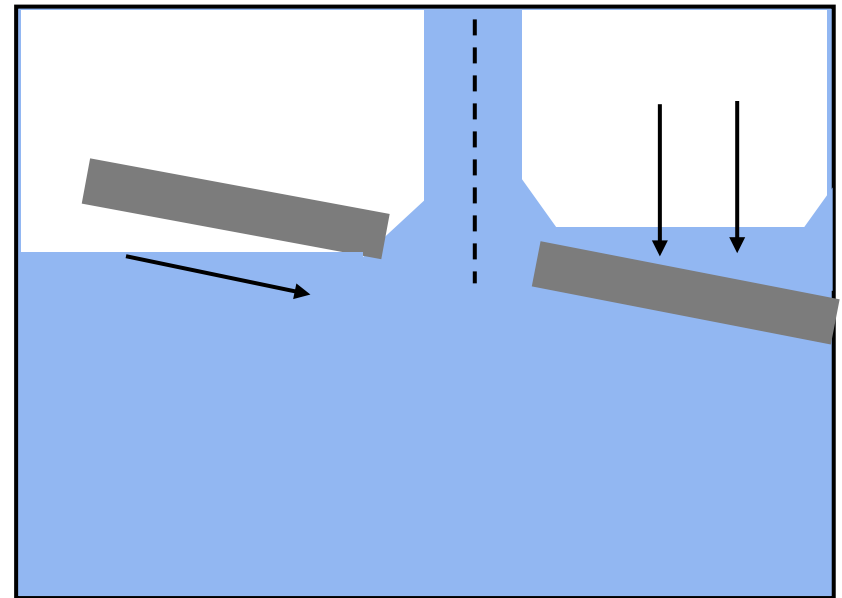
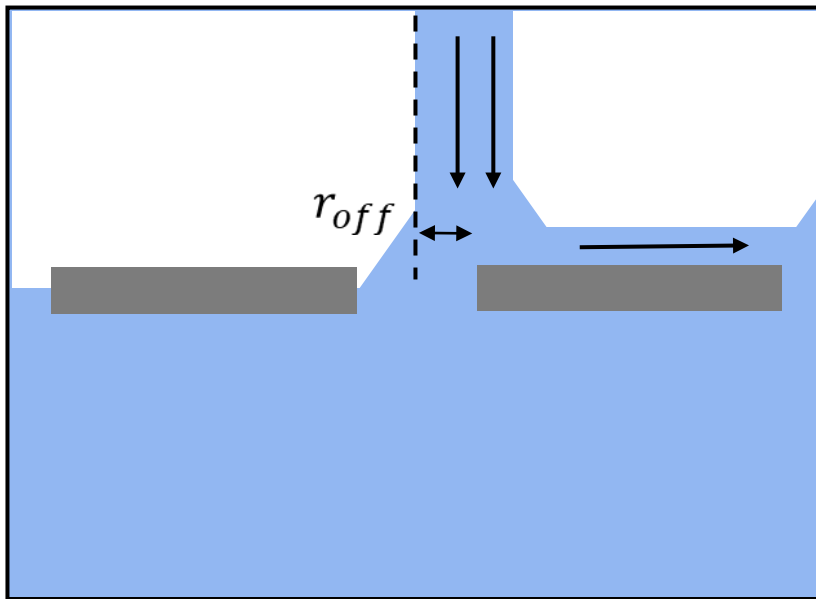


Solution %(Mass- Volume)	Temperat ure	Density	Viscosity	
	0	20	0.9982	1
	8	27	1.0559	1
	16	35	1.1162	1
	20	43	1.1478	1
	26	55	1.193	1

Appendix F: Stability Analysis



Jet does not collide with edges of disk



Uneven force distribution from un-centered flow causes torque on one side of disk.

Causes higher edge of disk to shift towards stream, correcting back towards steady state

Appendix G: Torque Calculation

From previous calculation of force colliding on disk:

$$F_a = v_1^2 \rho \pi r_{jet}^2 - v_2^2 \rho A_2$$

$$\Gamma = r_{off} \times F_a = r_{off} F_a \sin 90^\circ$$

If $\Gamma > \Gamma_b + \Gamma_s$, disk will tip over and sink

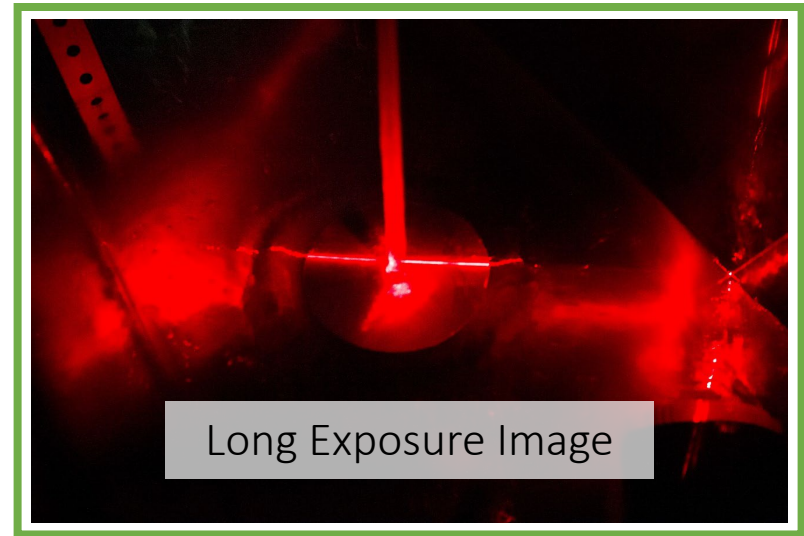
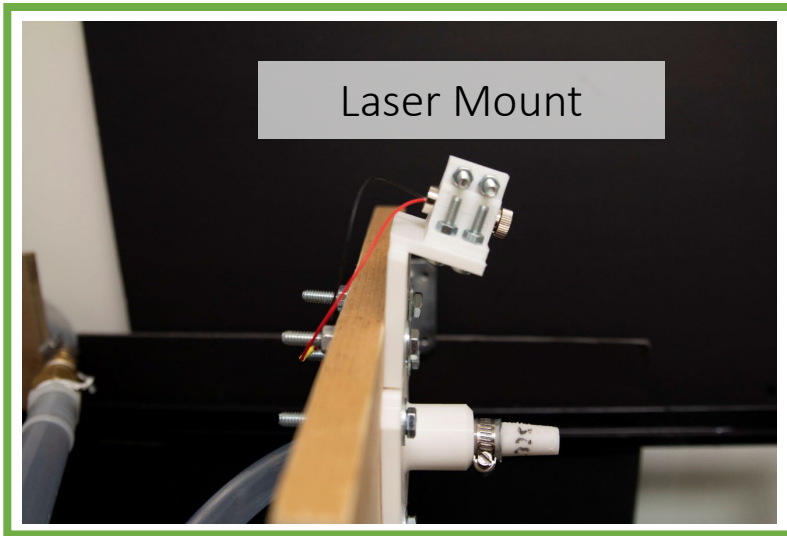
Critical condition:

$$r_{off} F_a \sin 90^\circ = R F_b \sin 90^\circ + R F_s \sin 90^\circ$$

r_{off} scales linearly with disk radius

$$r_{off} = \frac{R F_b + R F_s}{F_a}$$

Appendix H: Laser Mount



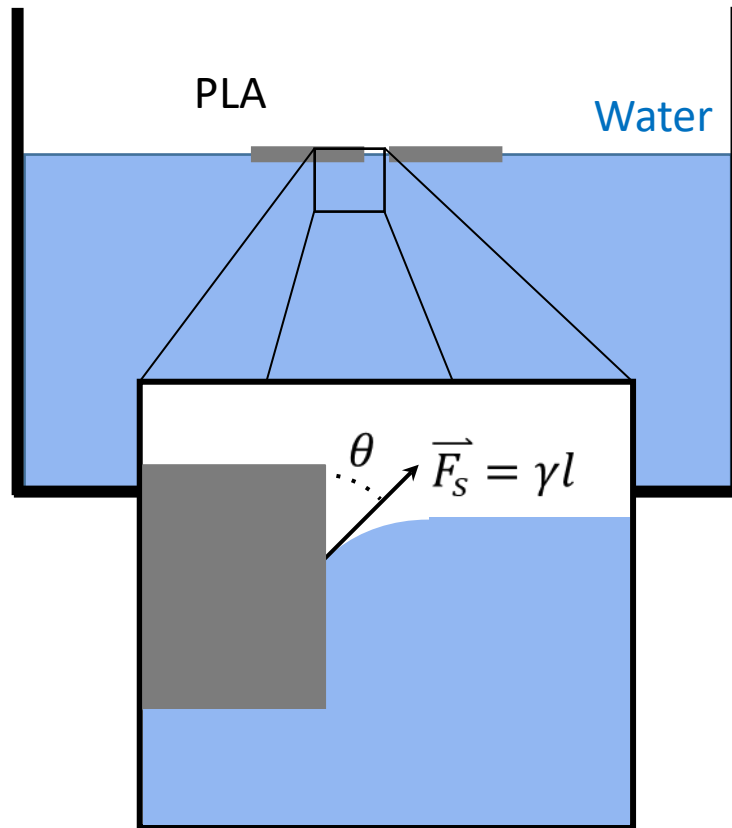
Specifications:

PLA printed 70 degree mount

Long Exposure Images remove turbulence from image

To measure radius and visualize hydraulic jump

Appendix I: Surface Tension, Density, Pressure and Viscosity



From literature, @ 20.9°C:

$$\gamma = 0.0435 \frac{\text{N}}{\text{m}}$$

$$\theta = 67.7^\circ$$

(Biresaw & Carriere, 2001)

$$\rho = 998.2 \frac{\text{kg}}{\text{m}^3}$$

(Simetric, 2015)

$$P_{atm} = 101.325 \text{ kPa}$$

(Engineering Toolbox, 2004)

$$\mu = 0.9775 \cdot 10^{-3} \text{ Pa} \cdot \text{s}$$

(IAPWS, 2008)

Appendix K: Bernoulli Assumptions



Inviscid Fluid



Steady fluid flow



Fluid is incompressible

Appendix L: Calculating Reynolds Number

$$Re = \frac{\rho v L}{\mu}$$

$\rho = \text{density}$

$v = \text{flow speed}$

$L = \text{characteristic linear dimension}$

$\mu = \text{dynamic viscosity of the fluid}$

$Re: 25000 - 35000$

Appendix M: Measuring Physical

F



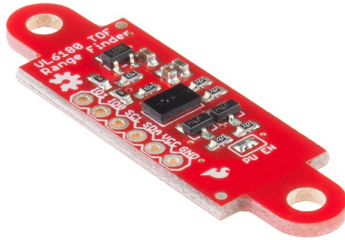
Flow Meter $\pm 0.7\%$



Analytical Balance $\pm 0.01g$



5kg Load Cell + HX711 Amplifier



VL6180 Range Sensor: $\pm 0.5mm$



Digital Caliper $\pm 0.02mm$



Canon EOS 100D – DSLR Camera